

ANALYSIS OF SHIP FLOW IN AN
IDEAL FLUID USING GUILLOTON'S
METHOD AND SPLINE FUNCTIONS

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ANALYSIS OF SHIP FLOW IN AN IDEAL FLUID
USING GUILLOTON'S METHOD AND SPLINE FUNCTIONS

BY



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ABSTRACT

In this thesis, the numerical evaluation is formulated for computing the linearized disturbance velocity of a steady, inviscid free surface gravity flow past a ship hull. The hull is represented by a system of source panels with uniformly distributed strengths on the centerplane. The improvement of the results on the boundaries, i.e. free surface and hull surface, by Guilloton's method is investigated. Based on Guilloton's method, thin-ship-panel approximation and cubic spline curve fitting, a scheme has been developed for setting up a computer program to compute the ship wave-making resistance, flow around the ship hull and wave elevation along the ship side. The results of sample calculations for standard hull forms of Wigley model 3012 and Series 60 block 60 have shown good agreement with the experimental results for Froude numbers between 0.25 and 0.35 which are just in the speed range of the conventional merchant ships.

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NOMENCLATURE

Linearized space (LS)	= The domain (x_o, y_o, z_o) of the linearized solution.
Real space (RS)	= The domain (x, y, z) of the Guilloton's solution.
$f(x, z), f^0(x_o, z_o)$	= Ship hull functions in RS and LS, respectively.
$\zeta(x, y), \zeta^0(x_o, y_o)$	= Wave elevations in RS and LS respectively.
R, R_o	= Wave-making resistances in RS and LS, respectively.
g	= Gravitational acceleration.
$\vec{q}(x, y, z)$	= (u, v, w) , Disturbance velocity vector.
ϵ	= Perturbation parameter.
$G(x, y, z; x', 0, z')$	= Green's function of linearized problem.
Re	= Real part of complex function.
Im	= Imaginary part of complex function.
C_1, C_2	= Integrating paths in complex plane.
$m(x', z')$	= Strength of source distribution on the centerplane.
u_{ij}, v_{ij}, w_{ij}	= Element velocity components induced by a source panel (i, j) on the centerplane.
R_{ij}	= Element wave-making resistance induced by a source panel (i, j) on the centerplane.
ζ_{ij}	= Element wave-elevation induced by a source panel (i, j) on the centerplane.

I_1, I_2, S, I_R = Complex integrals

$\text{Res}(a)$ = Residue of complex function, $f(k)$, at pole $k = a$

$\text{sig } w$ = -1 for $w < 0$, 1 for $w > 0$

$H(x - x')$ = Heaviside "unit step function",
0 for $x < x'$, 1 for $x > x'$

$$\{f(x', z')\} \begin{vmatrix} x'_i+1 \\ x'_i \end{vmatrix} \begin{vmatrix} z'_j+1 \\ z'_j \end{vmatrix} = \{f(x', z')\} \begin{matrix} x' = x'_i+1 \\ z' = z'_j+1 \end{matrix} - \{f(x', z')\} \begin{matrix} x' = x'_i \\ z' = z'_j \end{matrix}$$

$E_1(\lambda) = \int_{-\infty}^{\infty} \frac{e^{-K}}{K} dK$, (λ is a complex number), or
called complex exponential integral.

n or ETA = Nondimensional wave elevation.

$A(x_o, z_o), B(x_o, z_o), C(x_o, z_o)$ = Displacements of Guilloton's transformation
in x_o -, y_o - and z_o - directions, respectively.

F.P. = Forward perpendicular of a ship (-1)

A.P. = After perpendicular of a ship (1)

C_w = $Rw/(1/2 \rho S U^2)$, Wave resistance coefficient

L = $1/2$ LBP

CHAPTER 1

INTRODUCTION

Since Michell [1] developed the thin ship theory to solve the linearized problem of the waves produced by a ship of given form moving with uniform velocity in the free surface of unbounded water which is considered to be inviscid, many researchers have tried to modify the thin ship theory to obtain more reasonable results by including the nonlinear effects. From a practical point of view, one of the notable methods was developed by Guilloton [2] based on geometrical and intuitive physical reasoning and was formulated in mathematical form by Gadd [10]. As a matter of fact, Guilloton's basic idea is the same as that of the well-known "strained coordinates method" which was developed by Poincaré and successfully used in some singular perturbation problems [3]. The main idea of this kind of method is that the linearized solution of the nonlinear problem may have the right form, but not quite at the right place, so that the remedy is to slightly strain the coordinates or set up a transformation between the "linearized space" and the "real space". Following the method of strained coordinates, a perturbation analysis can be carried out to rationalize Guilloton's method, such as in [4] and [5]. They have shown that Guilloton's solution is essentially equivalent to an inconsistent second order approximation, in which the field equation is satisfied to first order and the boundary conditions are satisfied to second order.

From the computational point of view, Guilloton's method includes the following three sub-procedures:

1. Find the "linearized hull" corresponding to a given "real hull" for a given Froude number by an "inverse Guilloton's transformation". This is an iterative process.
2. Calculate the flow quantities around the "linearized hull" by the thin ship theory.
3. Transform the calculated flow quantities to that around the "real hull" by Guilloton's transformation.

Based on these sub-procedures, a computer program has been developed to analyse the flow and isobar around the ship hull, the wave elevation along the ship side and the ship wave-making resistance. In mathematical principle, Guilloton's solution is still only of first-order accuracy. But from the results of numerical experiments, the prediction of the ship flow with Guilloton's method is much better than that with the thin ship theory for Froude numbers of about 0.25 to 0.35. The use of Guilloton's method for preliminary design of conventional ships has a good potential.

CHAPTER 2

THEORETICAL BACKGROUND AND NUMERICAL FORMULATION2.1 Exact and linearized steady ship-flow problem

It is assumed that the fluid is incompressible and inviscid and that the flow is irrotational. Let Oxyz be a moving coordinate system fixed on the ship, with velocity $\vec{U} = (-U, 0, 0)$ with respect to the obvious inertial frame and the origin O is at midship. The Oxy plane is taken to coincide with the undisturbed free-surface and the z-axis is positive upwards as in Figure 1. The effects of sinkage and trim are not considered. The hull equation is $y = f(x, z)$ for $-L \leq x \leq L$, $-b(x) \leq z \leq \zeta(x, y)$, where $2L$ is the length of the ship, $z = \zeta(x, y)$ denotes the elevation of the disturbed free surface and $z = -b(x)$ is the equation of the keel line. Now, the problem of evaluating the ship flow is reduced to determine the disturbance velocity potential, say, $\phi(x, y, z)$, which satisfies Laplace's equation, $\nabla^2 \phi = 0$, in the flow field with the following exact boundary conditions as given in [6]:

- (a) The kinematic boundary condition on the hull surface:

$$\phi_x(x, f(x, z), z) f_x(x, z) + \phi_y + \phi_z f_z = -U f_x \quad (1.1)$$

- (b) The kinematic boundary condition on the free surface:

$$\phi_x(x, y, \zeta(x, y)) \zeta_x(x, y) + \phi_y \zeta_y + \phi_z = -U \zeta_x \quad (1.2)$$

- (c) The dynamic boundary condition on the free surface:

$$U \phi_x(x, y, \zeta(x, y)) - 1/2(\phi_x^2 + \phi_y^2 + \phi_z^2) + g \zeta(x, y) = 0 \quad (1.3)$$

- (d) The kinematic boundary condition on the ocean bottom,
(if it is assumed to be infinite):

$$\phi_z(x, y, z) = 0 \quad \text{as} \quad z \rightarrow -\infty \quad (1.4)$$

- (e) The radiation condition specifying that waves are not propagated upstream from the ship but only downstream.

If the disturbance velocity potential ϕ can be found, then the disturbance velocity, $\dot{q}(x, y, z)$, the wave-making resistance, R , and the wave elevation, $\zeta(x, y)$, can be calculated as follows:

$$\dot{q}(x, y, z) = \nabla \phi(x, y, z) \quad (1.5)$$

$$R = \iint_{\text{hull}} p \cos(n, x) ds = 2 \iint_{S_0} p(x, f(x, z), z) f_x(x, z) dx dz$$

$$= -2p \iint_{S_0} [U \phi_x(x, f(x, z), z) + 1/2(\phi_x^2 + \phi_y^2 + \phi_z^2) + gz] f_x(x, z) dx dz \quad (1.6)$$

$$\text{and, } \zeta(x, y) = -\frac{1}{g} [U \phi_x(x, y, \zeta(x, y)) + 1/2(\phi_x^2 + \phi_y^2 + \phi_z^2)] \quad (1.7)$$

where S_0 is the projection of the wetted surface on the centerplane and p is the pressure on the ship hull.

The formula for the wave elevation is an implicit form since the right hand side of (1.7) is also a function of wave elevation.

The difficulty of this "exact" problem stems from the fact that the position of the free surface and the extent of the wetted area of the hull surface are initially unknown and are to be determined as part of the solution; also the boundary conditions are non-linear. One of the procedures for linearizing the problem begins by writing the equation of the ship hull in the form $y = \epsilon f(x, z)$ where ϵ is a beam-length ratio. It is assumed that the disturbance velocity potential, wave elevation and wave-making resistance can be expanded in power series of ϵ as follows:

$$\begin{aligned}\phi(x, y, z; \epsilon) &= \epsilon \phi^{(1)}(x, y, z) + \epsilon^2 \phi^{(2)} + \dots \\ \zeta(x, y; \epsilon) &= \epsilon \zeta^{(1)}(x, y, z) + \epsilon^2 \zeta^{(2)} + \dots \\ R(\epsilon) &= \epsilon^2 R^{(1)} + \epsilon^3 R^{(2)} + \dots\end{aligned}\quad (1.8)$$

The expansions (1.8) are now substituted into (1.1) to (1.4). After they have been expanded as a power series in ϵ , it is found that the velocity potential ϕ of the first order approximation must satisfy Laplace's equation together with the following linearized boundary conditions:

$$(a) \quad \phi_y(x, \pm 0, z) = \mp U f'_x(x, z) \quad (1.9)$$

$$(b) \quad \phi_z(x, y, 0) = U \zeta_x(x, y) \quad (1.10)$$

$$(c) \quad \zeta(x, y) = - \frac{U}{V} \phi_x(x, y, 0) \quad (1.11)$$

$$(d) \quad \phi_z(x, y, z) = 0 \quad \text{as} \quad z \rightarrow -\infty \quad (1.12)$$

(e) The radiation condition mentioned before.

The wave-making resistance is changed from (1.6) to the following form

$$R = -2\rho U \iint_{S_0} \phi_x(x, 0, z) f_x(x, z) dx dz \quad (1.13)$$

where S_0 is the ship centerplane for $-L \leq x \leq L$ and $-b(x) \leq z \leq 0$. The free-surface boundary conditions (1.10) and (1.11), are combined to give

$$\phi_{xx}(x, y, 0) + K_0 \phi_z = 0 \quad \text{where} \quad K_0 = \frac{g}{U^2} \quad (1.14)$$

2.2 Solution of linearized ship-flow problem

The linearized ship flow problem is a mathematical boundary-value problem which can be solved by the Green's function method [7], i.e. representing the body by a distribution of singularities. The linearized disturbance velocity potential can be expressed in the form:

$$\phi(x, y, z) = \iint_{S_0} 2U f_x(x', z') G(x, y, z; x', 0, z') dx' dz' \quad (2.1)$$

where

$G(x, y, z; x', 0, z')$ is Green's function or the unit source function

of the linearized problem.

$f_{x'}(x', z')$ is the longitudinal slope of ship hull. The "prime" system also denotes coordinates on the body.

In this linearized solution, equation (2.1) shows that the sources are distributed on the ship centerplane and the source strength is only dependent on the longitudinal slope of the ship hull. We define the source strength $m(x', z') = 2Uf_{x'}(x', z')$ and equation (2.1) becomes

$$\phi(x, y, z) = \iint_{S_0} m(x', z') G(x, y, z; x', 0, z') dx' dz' \quad (2.2)$$

Green's function of the linearized problem developed by Havelock, [8] with the image method is given in the following form

$$4\pi G(x, y, z; x', 0, z') = -\frac{1}{r_1} + \frac{1}{r_2} + \frac{iK_0 \operatorname{Re} \left[\int_{-\pi}^{\pi} \sec^2 \theta d\theta \int_0^\infty \frac{e^{K[(z+z')+i\omega k]}}{K-K_0 \sec^2 \theta} dk \right]}{\pi} \quad (2.3)$$

where

$$r_1 = [(x-x')^2 + y^2 + (z-z')^2]^{1/2}$$

$$r_2 = [(x-x')^2 + y^2 + (z+z')^2]^{1/2}$$

$$\omega = (x-x') \cos \theta + y \sin \theta$$

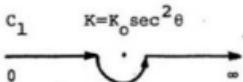
$$\text{and } K_0 = g/U^2$$

The double integral of Green's function cannot be determined without a statement on the procedure of integration around the singularity, $K = K_0 \sec^2 \theta$. That is to say that a way (as in [9]) should be chosen such that the free waves only trail behind the ship. The radiation condition mentioned above makes the solution to this mathematical problem unique.

Let

$$\begin{aligned}
 I &= R e \left\{ \int_{-\pi}^{\pi} \sec^2 \theta d\theta \int_0^{\infty} \frac{e^{K[(z+z')+i\omega]}}{K - K_0 \sec^2 \theta} dK \right\} \\
 &= R e \left\{ \int_{-\pi/2}^{\pi/2} \sec^2 \theta d\theta \left[\int_0^{\infty} \frac{e^{K[(z+z')+i\omega]}}{K - K_0 \sec^2 \theta} dK + \int_0^{\infty} \frac{e^{K[(z+z')-i\omega]}}{K - K_0 \sec^2 \theta} dK \right] \right\} \\
 &\quad (C_1) \qquad (C_2)
 \end{aligned} \tag{2.4}$$

The integration paths, C_1 and C_2 , have to be chosen as follows:



we define

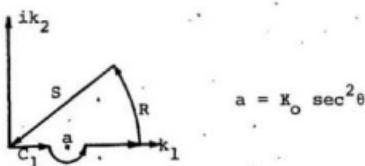
$$I_1 = \int_{C_1} \frac{e^{K[(z+z')+i\omega]}}{K - K_0 \sec^2 \theta} dK \quad \text{and} \quad I_2 = \int_{C_2} \frac{e^{K[(z+z')-i\omega]}}{K - K_0 \sec^2 \theta} dK$$

(2.5)

I_1 and I_2 can be treated by contour integration in the complex plane to obtain a nonoscillatory integrand.

(1) When $\omega > 0$

(a) Integral I_1



$$a = K_0 \sec^2 \theta$$

$$I_1 = 2\pi i \operatorname{Res}(a) - I_S \quad (\lim_{R \rightarrow \infty} I_R = 0) \quad (2.6)$$

$$2\pi i \operatorname{Res}(a) = 2\pi \left(e^{K_0 \sec^2 \theta (z+z')} \right) \left[-\sin(K_0 \sec^2 \theta \cdot \omega) + i \cos(K_0 \sec^2 \theta \cdot \omega) \right] \quad (2.7)$$

$$I_S = \int_S \frac{e^{K[(z+z') + i\omega]}}{K - K_0 \sec^2 \theta} dK \quad (2.8)$$

The path S can be chosen so as to make the argument of the exponential real along the path and hence eliminating the oscillatory behaviour of the integrand in I_S . So

$$\operatorname{Im} \{K[(z+z') + i\omega]\} = 0 \quad \text{and} \quad K = k_1 + ik_2 \quad (2.9)$$

$$k_2 = -\omega k_1 / (z+z') , \quad K = k_1 [(z+z') - i\omega] / (z+z') \quad (2.10)$$

Substituting (2.10) into (2.8)

$$I_S = \int_{-\infty}^0 \frac{e^{[(z+z')^2 + \omega^2] k_1 / (z+z')}}{[(z+z')^2 + \omega^2] k_1 / (z+z') - K_0 \sec^2 \theta [(z+z') + i\omega]} \cdot \frac{[(z+z')^2 + \omega^2]}{(z+z')} dk_1 \quad (2.11)$$

since $z+z' < 0$

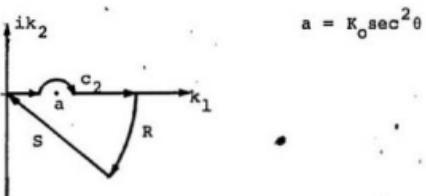
changing variable let $-K = [(z+z')^2 + \omega^2] k_1 / (z+z')$

$$I_S = - \int_0^\infty \frac{e^{-K}}{[K + K_0 \sec^2 \theta + (z+z')] + i(K_0 \sec^2 \theta \cdot \omega)} dK \quad (2.12)$$

Substituting (2.7) and (2.12) into (2.6) and taking the real part.

$$\begin{aligned} \operatorname{Re}(I_1) &= \int_0^\infty \frac{[K + K_0 \sec^2 \theta + (z+z')] e^{-K}}{[K + K_0 \sec^2 \theta + (z+z')]^2 + (K_0 \sec^2 \theta \cdot \omega)^2} dK \\ &= 2\pi e^{-K_0 \sec^2 \theta \cdot (z+z')} \sin(K_0 \sec^2 \theta \cdot \omega) \end{aligned} \quad (2.13)$$

(b) Integral I_2



$$I_2 = -2\pi i \operatorname{Res}(a) - I_S \quad (\lim_{R \rightarrow \infty} I_R = 0) \quad (2.14)$$

$$-2\pi i \operatorname{Res}(a) = 2\pi \left\{ e^{K_0 \sec^2 \theta \cdot (z+z')} \right. \\ \left[-\sin(K_0 \sec^2 \theta \cdot \omega) - i \cos(K_0 \sec^2 \theta \cdot \omega) \right] \quad (2.15)$$

$$I_S = \int_S \frac{e^{K[(z+z')-i\omega]}}{K-K_0 \sec^2 \theta} dK \quad (2.16)$$

Similarly,

$$I_m \{K[(z+z')-i\omega]\} = 0 \quad \text{and} \quad K = k_1 + ik_2 \quad (2.17)$$

$$k_2 = \omega k_1 / (z+z') \quad , \quad K = k_1 [(z+z') + i\omega] / (z+z') \quad (2.18)$$

Substituting (2.18) into (2.16).

$$I_S = \int_{-\infty}^0 \frac{e^{[(z+z')^2 + \omega^2]k_1/(z+z')}}{[(z+z')^2 + \omega^2]k_1/(z+z') - K_0 \sec^2 \theta [(z+z') - i\omega]} dk_1 \quad (2.19)$$

$$\frac{[(z+z')^2 + \omega^2]}{(z+z')} dk_1 \quad (2.19)$$

since $z+z' < 0$

changing variable let $-K = [(z+z')^2 + \omega^2]k_1/(z+z')$

$$I_S = - \int_{-\infty}^{\infty} \frac{e^{-K}}{[K+K_0 \sec^2 \theta \cdot (z+z')] - i(K_0 \sec^2 \theta \cdot \omega)} dK \quad (2.20)$$

Substituting (2.15) and (2.20) into (2.14) and taking the real part

$$\begin{aligned} \operatorname{Re}\{I_2\} &= \int_0^\infty \frac{[K+K_0 \sec^2 \theta \cdot (z+z')] e^{-K}}{[K+K_0 \sec^2 \theta \cdot (z+z')]^2 + (K_0 \sec^2 \theta \cdot \omega)^2} dK \\ &= 2\pi \cdot e^{K_0 \sec^2 \theta \cdot (z+z')} \cdot \sin(K_0 \sec^2 \theta \cdot \omega) \quad (2.21) \end{aligned}$$

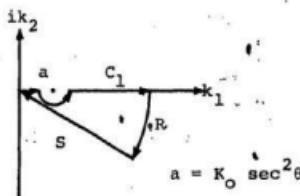
Substituting (2.13) and (2.21) into (2.4), the integral I for $\omega > 0$ is

$$\begin{aligned} I &= 2 \int_{-\pi/2}^{\pi/2} \sec^2 \theta d\theta \int_0^\infty \frac{[K+K_0 \sec^2 \theta \cdot (z+z')] e^{-K}}{[K+K_0 \sec^2 \theta \cdot (z+z')]^2 + (K_0 \sec^2 \theta \cdot \omega)^2} dK \\ &= 4\pi \int_{-\pi/2}^{\pi/2} \sec^2 \theta \cdot e^{K_0 \sec^2 \theta \cdot (z+z')} \cdot \sin(K_0 \sec^2 \theta \cdot \omega) d\theta \quad (2.22) \end{aligned}$$

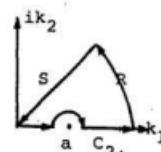
(2) When $\omega < 0$

Similarly, the contour integrations are chosen as

(a)



(b)



The following result for $\omega < 0$, can be obtained

$$I = 2 \int_{-\pi/2}^{\pi/2} \sec^2 \theta d\theta \int_0^\infty \frac{[K + K_0 \sec^2 \theta \cdot (z+z')] e^{-K}}{[K + K_0 \sec^2 \theta \cdot (z+z')]^2 + (K_0 \sec^2 \theta \cdot \omega)^2} dK \quad (2.23)$$

Substituting (2.22) and (2.23) into (2.3). Green's function becomes

$$\begin{aligned} 4\pi G(x, y, z; x', 0, z') &= -\frac{1}{r_1} + \frac{1}{r_2} \\ &+ 2K_0 \int_{-\pi/2}^{\pi/2} \sec^2 \theta d\theta \int_0^\infty \frac{[K + K_0 \sec^2 \theta \cdot (z+z')] e^{-K}}{[K + K_0 \sec^2 \theta \cdot (z+z')]^2 + (K_0 \sec^2 \theta \cdot \omega)^2} dK \\ -4K_0 \int_{-\pi/2}^{\pi/2} &\left(\frac{1+\text{sig}\omega}{2} \right) \sec^2 \theta \cdot e^{K_0 \sec^2 \theta \cdot (z+z')} \sin(K_0 \sec^2 \theta \cdot \omega) d\theta \end{aligned} \quad (2.24)$$

where

$$\text{sig } \omega = \begin{cases} -1 & \text{for } \omega < 0 \\ 1 & \text{for } \omega > 0 \end{cases}$$

Since most of the quantities of practical interests, such as wave-making resistance, wave elevation along the ship hull and flow around the ship hull can be obtained from flow variables evaluated at the centerplane, the linearized disturbance velocity

potential on the centerplane, $\phi(x, 0, z)$, associated with a given source distribution $m(x', z')$, is the most important quantity to be defined in the computational procedure:

$$\phi(x, 0, z) = \iint_{S_0} m(x', z') \cdot G(x, 0, z; x', 0, z') dx' dz' \quad (2.25)$$

with

$$4\pi G(x, 0, z; x', 0, z') = -\frac{1}{r_1} + \frac{1}{r_2} + \frac{4K_0}{\pi} \int_0^{\pi/2} \sec^2 \theta d\theta \int_0^\infty \frac{[K + K_0 \sec^2 \theta \cdot (z+z')] e^{-K}}{[K + K_0 \sec^2 \theta \cdot (z+z')]^2 + (K_0 \sec^2 \theta \cdot \omega)^2} dK$$

$$-H(x-x') \frac{8K_0}{\int_0^{\pi/2} \sec^2 \theta \cdot e^{K_0 \sec^2 \theta \cdot (z+z')} \sin(K_0 \sec^2 \theta \cdot \omega) d\theta} \quad (2.26)$$

where

$$r_1 = [(x-x')^2 + (z-z')^2]^{1/2}$$

$$r_2 = [(x-x')^2 + (z+z')^2]^{1/2}$$

$$\omega = (x-x') \cos \theta$$

and $H(x-x')$ is the Heaviside "unit step function" which is defined as

$$H(x-x') = \begin{cases} 0 & \text{for } x < x' \\ 1 & \text{for } x > x' \end{cases}$$

Substituting (2.25) and (2.26) into the resistance formula (1.13), only the last integral in the expression for G leads to a nonzero term. This gives the well-known Michell's integral of ship wave-making resistance:

$$R = \frac{\rho K_0^2}{\pi} \int_{-\pi/2}^{\pi/2} [P(\theta)^2 + Q(\theta)^2] \sec^3 \theta d\theta \quad (2.27)$$

with $P(\theta) = \iint_{S_0} m(x', z') e^{K_0 \sec^2 \theta \cdot z'} \cos(K_0 \sec \theta \cdot x') dx' dz'$

and $Q(\theta) = \iint_{S_0} m(x', z') e^{K_0 \sec^2 \theta \cdot z'} \sin(K_0 \sec \theta \cdot x') dx' dz'$

Similarly, substituting (2.25) and (2.26) into the wave elevation formula (1.11), the wave elevation along the ship hull can be written as

$$\zeta(x, 0) = - \frac{U}{g} \iint_{S_0} m(x', z') G_x(x, 0, 0; x', 0, z') dx' dz' \quad (2.28)$$

The disturbance velocity components around the ship hull can also be obtained in terms of Green's function and source distribution:

$$u(x, 0, z) = \phi_x(x, 0, z) = \iint_{S_0} m(x', z') G_x(x, 0, z; x', 0, z') dx' dz' \quad (2.29)$$

$$v(x, 0, z) = U f_x(x, z) = \frac{1}{2} m(x, z) \quad (2.30)$$

$$w(x, 0, z) = \phi_z(x, 0, z) = \iint_{S_0} m(x', z') G_z(x, 0, z; x', 0, z') dx' dz' \quad (2.31)$$

2.3 Thin-ship-panel approximation

In order to evaluate the numerical value of wave-making resistance (2.27), wave elevation along the ship hull (2.28), and disturbance velocity components around the ship hull (2.29), (2.30) and (2.31), the ship centerplane is discretized into a system of source panels with associated strengths, m_{ij} , $i = 1, 2, \dots, M$, $j = 1, 2, \dots, N$. It is assumed that the strength is uniformly distributed over each source panel, so that the strength in the above equations can be taken out of the integral sign for each source panel. Furthermore, Green's function (2.26) can be separated into three parts as

$$G(x, 0, z; x', 0, z') = G_1(x, 0, z; x', 0, z') + G_2(x, 0, z; x', 0, z') + G_3(x, 0, z; x', 0, z'), \quad (3.1)$$

where

$$G_1(x, 0, z; x', 0, z') = -\frac{1}{4\pi} \left(-\frac{1}{r_1} + \frac{1}{r_2} \right) \quad (3.2)$$

$$G_2(x, 0, z; x', 0, z')$$

$$= -\frac{K_0}{\pi^2} \int_0^{\pi/2} \sec^2 \theta d\theta \int_0^\infty \frac{[K + K_0 \sec^2 \theta \cdot (z+z')] e^{-K}}{[(K+K_0 \sec^2 \theta \cdot (z+z'))^2 + (K_0 \sec^2 \theta \cdot \omega)^2]} dK \quad (3.3)$$

$$G_3(x, 0, z; x', 0, z') = -H(x-x') \frac{2K_0}{\pi} \int_0^{\pi/2} \sec^2 \theta \cdot e^{-K_0 \sec^2 \theta \cdot (z+z')} \sin(K_0 \sec^2 \theta \cdot \omega) d\theta \quad (3.4)$$

for G_1 : contribution of the radical term

G_2 : contribution of the double integral term

G_3 : contribution of the single integral term

Let us first assume there is a single source panel (i,j) with unit strength below the undisturbed free surface in a uniform stream of velocity U in the direction of the positive x -axis and the coordinates of four corner points of this source panel are (x'_i, z_j) , (x'_{i+1}, z_j) , (x'_{i+1}, z'_{j+1}) and (x'_i, z'_{j+1}) respectively. Then the elemental quantities induced by this source panel are as follows:

- (1) The elemental disturbance velocity components, u_{ij} , v_{ij} and w_{ij} , on the centerplane.

From linearized boundary condition (1.9), once the source strength is given, the v -component velocity on the centerplane is determined. Only u -component and w -component velocities should be computed. We separate velocity components into three parts corresponding to the three parts of Green's function as in (3.1):

$$u_{ij}(x, 0, z) = u_1(x, 0, z) + u_2(x, 0, z) + u_3(x, 0, z) \quad (3.5)$$

$$w_{ij}(x, 0, z) = w_1(x, 0, z) + w_2(x, 0, z) + w_3(x, 0, z) \quad (3.6)$$

Equation (3.9) can be simplified as follows:

$$\text{Let } I_1 = \int_0^\infty e^{-K} \left[\log \left((K + K_0 \sec^2 \theta \cdot (z+z'))^2 + (K_0 \sec^2 \theta \cdot \omega)^2 \right) \right] dK \quad (3.10)$$

Integrating by parts

$$I_1 = 2 \log (K_0 \sec^2 \theta) + \log [(z+z')^2 + \omega^2] \\ + 2 \int_0^\infty \frac{[(K + K_0 \sec^2 \theta \cdot (z+z')) e^{-K}]}{[(K + K_0 \sec^2 \theta \cdot (z+z'))^2 + (K_0 \sec^2 \theta \cdot \omega)^2]} dK \quad (3.11)$$

The integral part of I_1 can be written in the complex form as

$$\text{Re} \left\{ \int_0^\infty \frac{e^{-K}}{[K + K_0 \sec^2 \theta \cdot (z+z')] + i(K_0 \sec^2 \theta \cdot \omega)} dK \right\} \\ = \text{Re} \left\{ e^{\lambda} \int_{-\lambda}^{\infty} \frac{e^{-K}}{K} dK \right\} \\ = \text{Re} \{ e^{\lambda} E_1(\lambda) \} \quad (3.12)$$

where

$E_1(\lambda)$ is a complex exponential integral

$$\text{and } \lambda = K_0 \sec^2 \theta [(z+z') + i\omega]$$

(a) The contribution of the radical term G_1

$$u_1(x, 0, z) = \int_{z_j}^{z_{j+1}} \int_{x_i}^{x_{i+1}} \frac{\partial G_1}{\partial x} dx' dz'$$

$$= \frac{1}{4\pi} \left\{ \log[(z'-x) + \sqrt{(x'-x)^2 + (z'-z)^2}] - \log[(z'+z) + \sqrt{(x'-x)^2 + (z'+z)^2}] \right\} \begin{vmatrix}_{x_i}^{x_{i+1}} \begin{vmatrix}_{z_j}^{z_{j+1}} \quad (3.7)$$

$$w_1(x, 0, z) = \int_{x_i}^{x_{i+1}} \int_{z_j}^{z_{j+1}} \frac{\partial G_1}{\partial z} dz' dx'$$

$$= \frac{1}{4\pi} \left\{ \log[(x'-x) + \sqrt{(x'-x)^2 + (z'-z)^2}] + \log[(x'-x) + \sqrt{(x'-x)^2 + (z'+z)^2}] \right\} \begin{vmatrix}_{z_j}^{z_{j+1}} \begin{vmatrix}_{x_i}^{x_{i+1}} \quad (3.8)$$

(b) The contribution of the double integral term G_2

$$u_2(x, 0, z) = \int_{z_j}^{z_{j+1}} \int_{x_i}^{x_{i+1}} \frac{\partial G_2}{\partial x} dx' dz'$$

$$= \frac{-1}{2\pi^2} \int_0^{\pi/2} d\theta \int_0^\infty e^{-K} \left\{ \log[(K + K_0 \sec^2 \theta (z+z'))^2 + (K_0 \sec^2 \theta \cdot \omega)^2] \right\}$$

$$\begin{vmatrix}_{x_i}^{x_{i+1}} \begin{vmatrix}_{z_j}^{z_{j+1}} dK \quad (3.9)$$

Substituting (3.11) and (3.12) into (3.9)

$$u_2(x, 0, z) = \frac{-1}{2\pi^2} \int_0^{\pi/2} [2 \log(K_0 \sec^2 \theta) + \log((z+z')^2 + \omega^2) \\ + 2 \operatorname{Re}[e^{\lambda} E_1(\lambda)]] \begin{vmatrix} x_{i+1} & z_{j+1} \\ x_i & z_j \end{vmatrix} dK \quad (3.13)$$

The first term of the integral is independent of x' and z' , so it can be eliminated. Equation (3.13) becomes

$$u_2(x, 0, z) = \frac{-1}{2\pi^2} \int_0^{\pi/2} [\log((z+z')^2 + \omega^2) + 2 \operatorname{Re}[e^{\lambda} E_1(\lambda)] \begin{vmatrix} x_{i+1} & z_{j+1} \\ x_i & z_j \end{vmatrix} dK \quad (3.14)$$

$$w_2(x, 0, z) = \int_{x_i}^{x_{i+1}} \int_{z_j}^{z_{j+1}} \frac{\partial G_2}{\partial z} dz' dx' \\ = \frac{-1}{\pi^2} \int_0^{\pi/2} \sec \theta d\theta \int_0^\infty e^{-K} \left[\tan^{-1} \left[\frac{K_0 \sec^2 \theta + \omega}{K + K_0 \sec^2 \theta + (z+z')} \right] \right] \begin{vmatrix} z_{j+1} & x_{i+1} \\ z_j & x_i \end{vmatrix} dK \quad (3.15)$$

Similarly, let

$$I_1 = \int_0^\infty e^{-K} \left[\tan^{-1} \left[\frac{K_0 \sec^2 \theta + \omega}{K + K_0 \sec^2 \theta + (z+z')} \right] \right] dK \quad (3.16)$$

Since $z+z' < 0$, there is a discontinuity at $K = -K_0 \sec^2 \theta$.

($z+z'$) in the integrand, the integral (3.16) has to be separated into two parts:

$$I_1 = \int_0^a e^{-K} \cdot \tan^{-1} \left[\frac{K_0 \sec^2 \theta \cdot \omega}{K-a} \right] dK + \int_a^\infty e^{-K} \cdot \tan^{-1} \left[\frac{K_0 \sec^2 \theta \cdot \omega}{K-a} \right] dK \quad (3.17)$$

$$\text{where } a = -K_0 \sec^2 \theta \cdot (z+z')$$

Integrating by parts

$$I_1 = (\text{sig } \omega) \pi \cdot e^{K_0 \sec^2 \theta \cdot (z+z')} + \tan^{-1} \left(\frac{\omega}{z+z'} \right) + \int_0^\infty \frac{(-K_0 \sec^2 \theta \cdot \omega) e^{-K}}{[K+K_0 \sec^2 \theta \cdot (z+z')]^2 + (K_0 \sec^2 \theta \cdot \omega)^2} dK \quad (3.18)$$

$$\text{where } \text{sig } \omega = \begin{cases} -1 & \text{for } \omega < 0 \\ 1 & \text{for } \omega > 0 \end{cases}$$

The integral part of I_1 can also be written in the complex form as

$$\begin{aligned} & \text{Im} \left\{ \int_0^\infty \frac{e^{-K}}{[K+K_0 \sec^2 \theta \cdot (z+z')] + i(K_0 \sec^2 \theta \cdot \omega)} dK \right\} \\ &= \text{Im} \{ e^{\lambda} \int_{-\lambda}^{\infty} \frac{e^{-K}}{K} dK \} \\ &= \text{Im} \{ e^{\lambda} E_1(\lambda) \} \quad (3.19) \end{aligned}$$

where

$$\lambda = K_0 \sec^2 \theta [(z+z') + i\omega]$$

Substituting (3.18) and (3.19) into (3.15)

$$w_2(x, 0, z) = \frac{-1}{\pi^2} \int_0^{\pi/2} \sec \theta \{ (\text{sig} w) \pi \cdot e^{K_o \sec^2 \theta (z+z')} \\ + \tan^{-1} \left(\frac{w}{z+z'} \right) + \text{Im}[e^{\lambda} E_1(\lambda)] \} \int_{z'_j}^{z'_{j+1}} \int_{x'_i}^{x'_{i+1}} d\theta \quad (3.20)$$

(c) The contribution of the single integral term G_3

$$u_3(x, 0, z) = \int_{z'_j}^{z'_{j+1}} \int_{x'_i}^{x'_{i+1}} \frac{\partial G_3}{\partial x} dx' dz' \\ = H(x-x') \cdot \frac{2}{\pi} \int_0^{\pi/2} e^{K_o \sec^2 \theta \cdot (z+z')} \cdot \sin[K_o \sec \theta \cdot (x-x')] \quad (3.21)$$

$$\int_{x'_i}^{x'_{i+1}} \int_{z'_j}^{z'_{j+1}} d\theta \quad (3.21)$$

$$w_3(x, 0, z) = \int_{x'_i}^{x'_{i+1}} \int_{z'_j}^{z'_{j+1}} \frac{\partial G_3}{\partial z} dx' dz' \\ = -H(x-x') \cdot \frac{2}{\pi} \int_0^{\pi/2} e^{K_o \sec^2 \theta \cdot (z+z')} \cdot \cos[K_o \sec \theta \cdot (x-x')] \cdot \sec \theta \quad (3.22)$$

$$\int_{x'_i}^{x'_{i+1}} \int_{z'_j}^{z'_{j+1}} d\theta \quad (3.22)$$

(2) The elemental wave-making resistance R_{ij}

$$R_{ij} = \frac{\rho K_0^2}{\pi} \int_0^{\pi/2} [P_{ij}^2(\theta) + Q_{ij}^2(\theta)] \sec^3 \theta d\theta \quad (3.23)$$

where

$$P_{ij}(\theta) = \int_{z'_j}^{z'_j+1} \int_{x'_i}^{x'_i+1} e^{K_0 \sec^2 \theta \cdot z'} \cos(K_0 \sec \theta \cdot x') dx' dz'$$

$$= \frac{1}{K_0^2 \sec^3 \theta} \left((e^{K_0 \sec^2 \theta \cdot z'_j+1} - e^{K_0 \sec^2 \theta \cdot z'_j}) \right.$$

$$\left. \cdot [\sin(K_0 \sec \theta \cdot x'_i+1) - \sin(K_0 \sec \theta \cdot x'_i)] \right)$$

$$Q_{ij}(\theta) = \int_{z'_j}^{z'_j+1} \int_{x'_i}^{x'_i+1} e^{K_0 \sec^2 \theta \cdot z'} \sin(K_0 \sec \theta \cdot x') dx' dz'$$

$$= \frac{-1}{K_0^2 \sec^3 \theta} \left((e^{K_0 \sec^2 \theta \cdot z'_j+1} - e^{K_0 \sec^2 \theta \cdot z'_j}) \right.$$

$$\left. \cdot [\cos(K_0 \sec \theta \cdot x'_i+1) - \cos(K_0 \sec \theta \cdot x'_i)] \right)$$

Let

$$P_{ij}(\theta) = \frac{1}{K_0^2 \sec^3 \theta} \bar{P}_{ij}(\theta) \quad \text{and} \quad Q_{ij}(\theta) = \frac{-1}{K_0^2 \sec^3 \theta} \bar{Q}_{ij}(\theta)$$

The elemental wave-making resistance becomes

$$R_{ij} = \frac{\rho}{\pi K_0^2} \int_0^{\pi/2} [\bar{P}_{ij}^2(\theta) + \bar{Q}_{ij}^2(\theta)] \cos^3 \theta d\theta \quad (3.24)$$

where

$$\bar{p}_{ij}(\theta) = (e^{K_o \sec^2 \theta \cdot z'_{j+1}} - e^{K_o \sec^2 \theta \cdot z'_j}) [\sin(K_o \sec \theta \cdot x'_{i+1}) \\ - \sin(K_o \sec \theta \cdot x'_i)]$$

$$\bar{q}_{ij}(\theta) = (e^{K_o \sec^2 \theta \cdot z'_{j+1}} - e^{K_o \sec^2 \theta \cdot z'_j}) [\cos(K_o \sec \theta \cdot x'_{i+1}) \\ - \cos(K_o \sec \theta \cdot x'_i)]$$

(3) The elemental wave elevation $\zeta_{ij}(x, 0, 0)$

$$\zeta_{ij}(x, 0, 0) = - \frac{U}{g} \int_{z'_j}^{z'_{j+1}} \int_{x'_i}^{x'_{i+1}} \frac{\partial G}{\partial x'} dx' dz' = - \frac{U}{g} u_{ij}(x, 0, 0) \quad (3.25)$$

The total quantities induced by a system of source panels (i, j) $i=1, 2, \dots, M$, $j=1, 2, \dots, N$, associated with a system of strength m_{ij} , $i=1, 2, \dots, M$, $j=1, 2, \dots, N$ are the summation of the elemental quantities induced by each source panel as follows:

- (a) The disturbance velocity components $u(x, 0, z)$, $v(x, 0, z)$ and $w(x, 0, z)$

$$u(x, 0, z) = \sum_{j=1}^N \sum_{i=1}^M m_{ij} u_{ij}(x, 0, z) \quad (3.26)$$

$$v(x, 0, z) = U f_x(x, z) = m(x, z)/2 \quad (3.27)$$

$$w(x, 0, z) = \sum_{j=1}^N \sum_{i=1}^M m_{ij} w_{ij}(x, 0, z) \quad (3.28)$$

(b) The wave-making resistance R

$$R = \frac{\rho}{\pi K_0^2} \int_0^{\pi/2} [\bar{P}^2(\theta) + \bar{Q}^2(\theta)] \cos^3 \theta d\theta \quad (3.29)$$

where

$$\bar{P}(\theta) = \sum_{j=1}^N \sum_{i=1}^M m_{ij} \bar{P}_{ij}(\theta)$$

$$\bar{Q}(\theta) = \sum_{j=1}^N \sum_{i=1}^M m_{ij} \bar{Q}_{ij}(\theta)$$

(c) The wave elevation $\zeta(x, 0, 0)$

$$\zeta(x, 0, 0) = - \frac{U}{g} \sum_{j=1}^N \sum_{i=1}^M m_{ij} u_{ij}(x, 0, 0) = - \frac{U}{g} u(x, 0, 0) \quad (3.30)$$

Defining the nondimensional wave elevation/ $n = \zeta / (\frac{U^2}{2g})$, then

$$n(x, 0, 0) = -2 \frac{u(x, 0, 0)}{U} \quad (3.31)$$

2.4 Guilloton's Transformation:

Since in the thin ship theory, flow conditions on the hull surface, for example at a point P_1 in Fig. 2, are actually evaluated at P_o on the centerplane, Guilloton argues that if a transformation is to be applied to the y_o -direction in this way, similar transformations may be applied to the x_o - and z_o -directions. It is assumed that the time taken by a fluid particle to traverse the isobar from the forward perpendicular (F.P.) to a point $P(x,y,z)$ on the ship hull in the real space (RS) is approximately equal to x_o/U , the linearized estimate of flow time from F.P. to the corresponding point $P_o(x_o,0,z_o)$ in the linearized space (LS), where the flow velocity evaluated at the point P_o should apply at the point P (as in Fig. 2)[10].

So

$$dt = dx_o/U \text{ in LS}$$

$$dt = ds/(U + u(x,y,z)) = ds/(U + u_o(x_o,0,z_o)) \text{ in RS}$$

$$\text{for } ds = [1 + (\frac{\partial y}{\partial x})^2 + (\frac{\partial z}{\partial x})^2]^{1/2} dx \approx (1 + \alpha^2/2) dx$$

where $\alpha = \frac{\partial y}{\partial x}$, s is the distance along the isobar.

Hence, the transformation in the x_o -direction is

$$x = x_o + \int_{\text{F.P.}}^{x_o} \frac{u_o(x_o,0,z_o)/U - \alpha^2/2}{1 + \alpha^2/2} dx_o \quad (4.1)$$

The transformation in the y_o -direction is the same as that in the thin ship theory.

$$y = y_o + \int_{F.P.}^{x_o} \frac{m(x_o, z_o)}{2U} dx_o \quad (y_o = 0) \quad (4.2)$$

Since the fluid particle should be on the isobar, the transformation in the z_o -direction (refer to (1.11)) is

$$z = z_o - \frac{U}{g} u_o(x_o, 0, z_o) \quad (4.3)$$

From the geometrical point of view, the mapping function (4.1) maps a horizontal straight line on the centerplane in LS onto a space curve, the isobar along the ship hull, in RS. The same mapping relation will transform the prescribed values of velocity components, u_o , v_o , and w_o , (not velocity potential as in conformal mapping), along the straight line to corresponding values at points along the isobar.

So

$$u(x, y, z) = u_o(x_o, 0, z_o) \quad (4.4)$$

$$v(x, y, z) = v_o(x_o, 0, z_o) \quad (4.5)$$

$$w(x, y, z) = w_o(x_o, 0, z_o) \quad (4.6)$$

The wave elevations, $\zeta(x, y)$ and $\zeta^o(x_o, y_o)$, are defined on the undisturbed free surfaces, $z=0$ and $z_o=0$, in RS and LS respectively. So a mapping function for these two undisturbed free surfaces has only two components, $x = x(x_o, y_o)$ and $y = y(x_o, y_o)$. The wave elevation in RS can be written as

$$\zeta(x, y) = \zeta[x(x_o, y_o), y(x_o, y_o)] = \zeta^o(x_o, y_o) \quad (4.7)$$

That means the wave elevation at the point (x, y) in RS is the same as that at the corresponding point (x_o, y_o) in LS.

Similarly, the hull functions, $f(x, z)$ and $f^o(x_o, z_o)$, are defined on the centerplanes, $y=0$ and $y_o=0$, in RS and LS respectively. The hull function in RS can be written as

$$f(x, z) = f[x(x_o, z_o), z(x_o, z_o)] = f^o(x_o, z_o) \quad (4.8)$$

That means the breadth of the ship hull at point (x, z) in RS is the same as that at the corresponding point (x_o, z_o) in LS.

In fact, we can prove that this kind of transformation may make the solution in LS satisfy the exact kinematic boundary conditions, (1.1) and (1.2), in RS.

The exact and linearized kinematic boundary conditions on the free surface, (1.2) and (1.10), can be written as

$$\left[1 + \frac{u(x, y, \zeta(x, y))}{U}\right] \frac{\partial \zeta}{\partial x} + \frac{v(x, y, \zeta(x, y))}{U} \frac{\partial \zeta}{\partial y} = \frac{w(x, y, \zeta(x, y))}{U} \quad (4.9)$$

and

$$\frac{\partial \zeta^o}{\partial x_o} = \frac{w_o(x_o, y_o, 0)}{U} \quad (4.10)$$

Since the mapping function between the two undisturbed free surfaces is $x = x(x_o, y_o)$ and $y = y(x_o, y_o)$ and the wave elevations at the corresponding points, (x, y) and (x_o, y_o) ,

are the same, equation (4.10) can be written as

$$\frac{\partial x}{\partial x_0} \frac{\partial z}{\partial x} + \frac{\partial y}{\partial x_0} \frac{\partial z}{\partial y} = \frac{w_o(x_0, y_0, 0)}{U} \quad (4.11)$$

Comparing (4.11) and (4.9), we may take

$$u(x, y, z(x, y)) = u_o(x_0, y_0, 0) \quad (4.12)$$

$$v(x, y, z(x, y)) = v_o(x_0, y_0, 0) \quad (4.13)$$

$$w(x, y, z(x, y)) = w_o(x_0, y_0, 0) \quad (4.14)$$

and equation (4.9) will become

$$\left[1 + \frac{u_o(x_0, y_0, 0)}{U}\right] \frac{\partial z}{\partial x} + \frac{v_o(x_0, y_0, 0)}{U} \frac{\partial z}{\partial y} = \frac{w_o(x_0, y_0, 0)}{U} \quad (4.15)$$

Then the mapping function between the two undisturbed free surfaces may be written as

$$x = x_0 + \int_{-\infty}^{x_0} \frac{u_o(x_0, y_0, 0)}{U} dx_0 \quad (4.16)$$

$$y = y_0 + \int_{-\infty}^{x_0} \frac{v_o(x_0, y_0, 0)}{U} dx_0 \quad (4.17)$$

If the transformation of the velocity components, (4.12) to (4.14), is considered the transformation in the z_0 -direction can be found from (4.10) as

$$z = z_o + \int_{-\infty}^{x_o} \frac{w_o(x_o, y_o, 0)}{U} dx_o, \quad (z_o = 0) \quad (4.18)$$

From (1.11), (4.12) and (4.18), the dynamic boundary condition on the free surface in RS satisfied by this transformation is

$$\zeta(x, y) = -\frac{U}{g} u(x, y, \zeta(x, y)) \quad (4.19)$$

Although equation (4.19) is not the exact dynamic boundary condition (see (1.7)), it is a great improvement.

Similarly, in order to make the linearized solution satisfy the exact kinematic boundary condition on the hull surface in RS, the other set of transformations can be found as

$$x = x_o + \int_{-\infty}^{x_o} \frac{u_o(x_o, 0, z_o)}{U} dx_o \quad (4.20)$$

$$y = y_o + \int_{-\infty}^{x_o} \frac{v_o(x_o, 0, z_o)}{U} dx_o \quad (4.21)$$

$$z = z_o + \int_{-\infty}^{x_o} \frac{w_o(x_o, 0, z_o)}{U} dx_o \quad (4.22)$$

and

$$u(x, f(x, z), z) = u_o(x_o, 0, z_o) \quad (4.23)$$

$$v(x, f(x, z), z) = v_o(x_o, 0, z_o) \quad (4.24)$$

$$w(x, f(x, z), z) = w_o(x_o, 0, z_o) \quad (4.25)$$

Comparing (4.20) to (4.25) with (4.12) to (4.18), it can be shown that these two sets of transformation are consistent. The transformation for the whole flow field can be written as

$$x = x_0 + \int_{-\infty}^{x_0} \frac{u_o(x_0, y_0, z_0)}{U} dx_0 \quad (4.26)$$

$$y = y_0 + \int_{-\infty}^{x_0} \frac{v_o(x_0, y_0, z_0)}{U} dx_0 \quad (4.27)$$

$$z = z_0 + \int_{-\infty}^{x_0} \frac{w_o(x_0, y_0, z_0)}{U} dx_0 \quad (4.28)$$

$$u(x, y, z) = u_o(x_0, y_0, z_0) \quad (4.29)$$

$$v(x, y, z) = v_o(x_0, y_0, z_0) \quad (4.30)$$

$$w(x, y, z) = w_o(x_0, y_0, z_0) \quad (4.31)$$

and

$$\zeta(x, y) = \zeta^o(x_0, y_0) \quad (4.32)$$

$$f(x, z) = f^o(x_0, z_0) \quad (4.33)$$

Since only the ship hull in RS is known, the transformation (4.26) to (4.33), can be interpreted as a kind of inverse streamline tracing method which forces the ship hull in RS to be a stream surface. But obviously, the velocity components cannot satisfy the continuity equation in RS. (refer to conclusion)

Now, we assume that only the points on the centerplane in LS are considered; that the streamlines can be replaced by the isobars along the ship hull (recall the draft-length ratio of the ship is small); that the longitudinal shift can be calculated from the F.P.; and that the longitudinal slope of the ship hull is small. Then the transformation is found to be identical to that of Guilloton's method, (4.1) to (4.8).

In our problem, since the ship hull in RS and a specified Froude number are given, the "linearized hull" cannot be found directly. However, from the formulas of Guilloton's method, the source strength $m(x_0, z_0)$ in LS can be expressed in terms of the slopes, $\frac{\partial f}{\partial x}$ and $\frac{\partial f}{\partial z}$, of the given "real hull" as

$$\frac{m(x_0, z_0)}{2U} = \frac{1+u_0(x_0, 0, z_0)/U}{1+a^2/2} \frac{\partial f}{\partial x} - \frac{U^2}{g} \frac{a}{\partial x_0} \left(\frac{u_0(x_0, 0, z_0)}{U} \right) \frac{\partial f}{\partial z} \quad (4.34)$$

Therefore, the "linearized hull" can be found by the following iterative process:

- (1) Assume a trial value of the source strength for each source panel. The longitudinal local slope of the "real hull" corresponding to the centroid of each panel may be taken as the initial value of the quantity, $m/2U$, i.e. initially, the "linearized hull" is assumed to be the same as the given "real hull".

- (2) Calculate the u_o -component velocity (3.26) along each longitudinal straight line on the centerplane in LS.
- (3) Substitute the calculated u_o -component velocity into (4.1) and (4.3) to find the profile $\zeta(x, y)$ of each isobar in RS corresponding to the straight line in LS.
- (4) Calculate the slopes, $\frac{\partial f}{\partial x}$ and $\frac{\partial f}{\partial z}$, of the "real hull" corresponding to the points on each isobar and substitute them into (4.34) to find the new strength of each panel.
- (5) If the new strength of each source panel is the same as the trial value, the calculation stops. Otherwise, the new strength of each source panel will be taken as the new trial value and the calculation repeated.

After the "linearized hull" is found, the flow quantities can be calculated by the thin ship theory and transformed to that in RS. The wave-making resistance in RS actually should be calculated by integrating the pressure around the real hull as (1.6), but if we neglect the higher order term (as below), the wave-making resistance of the "linearized hull" can be taken as that of the "real hull".

The transformation of the points on the centerplane in LS can be written as

$$x = x_o + A(x_o, z_o) \quad A(x_o, z_o) = \int_0^{x_o} \frac{x_o u_o(x_o, 0, z_o)/U - a^2/2}{1 + a^2/2} dx_o \quad (4.35)$$

$$z = z_o + C(x_o, z_o) \quad C(x_o, z_o) = - \frac{U}{g} u_o(x_o, 0, z_o) \quad (4.36)$$

and equation (4.23) to (4.25)

According to Bernoulli's equation, the pressure at the point on the real hull is

$$P(x, f(x, z), z) = -\rho [U u(x, f(x, z), z) + \frac{1}{2} (u^2 + v^2 + w^2) + gz] \quad (4.37)$$

Substituting the corresponding linearized quantities into RHS of (4.37).

$$P(x, f(x, z), z) = -\rho [U u_0(x_0, 0, z_0) + \frac{1}{2} (u_0^2 + v_0^2 + w_0^2) + g[z_0 + C(x_0, z_0)]] \quad (4.38)$$

Expanding the longitudinal slope of the "real hull", $f_x(x, z)$, with respect to the linearized quantities, we can obtain:

$$f_x(x, z) = f_{x_0}^0(x_0, z_0) + (-f_{x_0}^0 A_{x_0} - f_{z_0}^0 C_{x_0}) + \dots \quad (4.39)$$

and

$$\begin{aligned} dx dz &= |x_{x_0} z_{z_0} - x_{z_0} z_{x_0}| dx_0 dz_0 \\ &= |1 + C_{z_0} A_{x_0} + A_{x_0} C_{z_0} - C_{x_0} A_{z_0}| dx_0 dz_0 \end{aligned} \quad (4.40)$$

Substituting (4.38), (4.39) and (4.40) into (1.6), the wave-making resistance of the real hull can be written as

$$\begin{aligned} R &= -2\rho U \iint_{S_0} u_0(x_0, 0, z_0) f_{x_0}^0(x_0, z_0) dx_0 dz_0 \\ &\quad - 2\rho \iint_{S_0} [U u_0(x_0, 0, z_0) + gC(x_0, z_0)] (C_{z_0} f_{x_0}^0 - C_{x_0} f_{z_0}^0) dx_0 dz_0 \quad (4.41) \\ &\quad + \dots \end{aligned}$$

The first term of (4.41) is the wave-making resistance, R_0 , of the linearized hull (see (1.13)), and since $C(x_0, z_0) = -\frac{U}{g} u_0(x_0, z_0)$, we find the second term is zero. Then the difference in the wave-making resistance between the "linearized hull" and the "real hull" is of the fourth order, i.e.

$$R = -2\rho U \iint_{S_0} u_0(x_0, 0, z_0) f_{x_0}^0(x_0, z_0) dx_0 dz_0 + O(\epsilon^4) \quad (4.42)$$

For certain nonlinear problems, we may find a transformation and then transform the linearized solution from the "linearized space" to the "real space". In this way, we may obtain a "better" solution. At least the solution satisfies the field equation and boundary conditions to the same order. But this is not the case for the three-dimensional steady ship-flow problem. Therefore the solution obtained from Guilloton's method can only be justified by comparing the computed results with the experimental results.

2.5 Cubic spline curve fitting[11]

Ordinarily, the shape of the ship hull is defined by the hull offsets. In order to find the slopes, $\frac{\partial f}{\partial x}$ and $\frac{\partial f}{\partial z}$, of the ship hull, we should find the equations of the smooth ship lines i.e. waterlines and framelines, passing through the given ship-offsets.

Let us consider a single-valued curve with continuous first and second derivatives passing through the given

points, say (x_i, y_i) , $i=1, 2, \dots, N$. This curve can be divided into $(N-1)$ segments corresponding to the given $(N-1)$ intervals. It is assumed that the equation of each segment is a three-degree polynomial, for example: the equation of i th segment is:

$$f_i(x) = A_i(x-x_i)^3 + B_i(x-x_i)^2 + C_i(x-x_i) + D_i \quad (5.1)$$

Hence, there are $4(N-1)$ unknowns, A_i , B_i , C_i and D_i , $i=1, 2, \dots, N-1$. We can find $4(N-1)$ equations from the following conditions to solve for these $4(N-1)$ unknowns:

- (a) The curve should pass through N given points (N equations)

$$f_i(x_i) = y_i, \quad i=1, 2, \dots, N-1 \quad (5.2)$$

$$f_{N-1}(x_N) = \tilde{y}_N \quad (5.3)$$

- (b) The curve is continuous, ($N-2$ equations)

$$f_i(x_{i+1}) = f_{i+1}(x_{i+1}) \quad i=1, 2, \dots, N-2 \quad (5.4)$$

- (c) The first derivative of the curve is continuous ($N-2$ equations)

$$f'_i(x_{i+1}) = f'_{i+1}(x_{i+1}) \quad i=1, 2, \dots, N-2 \quad (5.5)$$

- (d) The second derivative of the curve is continuous ($N-2$ equations)

$$f''_i(x_{i+1}) = f''_{i+1}(x_{i+1}) \quad i=1, 2, \dots, N-2 \quad (5.6)$$

- (e) Assume second derivatives of the ends are zero
(2 equations)

$$f''_1(x_1) = 0 \quad (5.7)$$

$$f_{N-1}''(x_N) = 0 \quad (5.8)$$

Solving this system of simultaneous equations is very tedious, after some manipulations, we can obtain the following formulas:

$$A_i = \frac{1}{6\Delta x_i} (y_{i+1}'' - y_i'')$$

$$B_i = \frac{1}{2} y_i'' \quad (5.9)$$

$$C_i = \frac{\Delta y_i}{\Delta x_i} - \frac{1}{6} \Delta x_i (y_{i+1}'' + 2y_i'')$$

$$D_i = y_i$$

and

$$\begin{bmatrix} 2(\Delta x_1 + \Delta x_2) & \Delta x_2 & & & \\ \Delta x_2 & 2(\Delta x_2 + \Delta x_3) & \Delta x_3 & & \\ & \Delta x_3 & 2(\Delta x_3 + \Delta x_4) & \Delta x_4 & \\ & & \ddots & & \\ & & \Delta x_{N-2} & 2(\Delta x_{N-2} + \Delta x_{N-1}) & \end{bmatrix} \begin{bmatrix} y_2 \\ y_3 \\ y_4 \\ \vdots \\ y_{N-1} \end{bmatrix}$$

$$\begin{aligned}
 & 6(\Delta y_2/\Delta x_2 - \Delta y_1/\Delta x_1) - \Delta x_1 y_1'' \\
 & 6(\Delta y_3/\Delta x_3 - \Delta y_2/\Delta x_2) \\
 & 6(\Delta y_4/\Delta x_4 - \Delta y_3/\Delta x_3) \\
 & \vdots \\
 & 6(\Delta y_{N-1}/\Delta x_{N-1} - \Delta y_{N-2}/\Delta x_{N-2}) - \Delta x_{N-1} y_N'' \\
 = & \boxed{\quad}
 \end{aligned} \tag{5.10}$$

where

$$\Delta x_i = x_{i+1} - x_i \quad \Delta y_i = y_{i+1} - y_i$$

$$y_i'' = f_i''(x_i) \quad y_{i+1}'' = f_{i+1}''(x_{i+1})$$

$$\text{and } y_i = f_i(x_i)$$

A special feature of equation (5.9) is that four coefficients of arbitrary segment i only depend on the two given points, (x_i, y_i) and (x_{i+1}, y_{i+1}) , and the two second derivatives, y_i'' and y_{i+1}'' , at given points. Hence, once equation (5.10) has been solved, the whole curve can be specified and the slope at any point of the curve can easily be calculated.

CHAPTER 3

COMPUTATIONAL METHOD AND RESULTS

We would like efficient computation and accurate results not only for wave-making resistance but also for flow quantities around the "real hull". The major differences in the computational scheme used from that of others were firstly using the source strength as the convergence criterion for finding the "linearized hull" and secondly using the cubic spline function for fitting the given ship-offsets. The computer programs were set up, taking into consideration the symmetric and antisymmetric properties of the element disturbance velocity components induced by the source panels. The whole computational scheme can be separated into two parts. One is to compute the disturbance velocity components, $u_o(x_o, 0, z_o)$ and $w_o(x_o, 0, z_o)$, induced by the source panels with "unit" strength, and the other is to find the source strength in LS as mentioned in section 2.4.

For the techniques used in computing the first part, we assume that the area of the centerplane of the "linearized hull" is the same as that of the "real hull". The centerplane is divided into 200 rectangular source panels (twenty-one stations including F.P. and A.P., and eleven waterlines including the base line and design waterline). The centroid of each panel is chosen as the control point (field point), so that there are 40,000 elemental disturbance velocities to be calculated,

i.e. the elemental disturbance velocity of each source panel with respect to each control point. It is a very tedious work. But as mentioned before, we can take advantage of symmetric and antisymmetric properties and the Heaviside function in the formulas of the elemental disturbance velocity components to reduce the laborious work considerably. For example: there are four source panels, A,B,C, and D in longitudinal direction such as

1	2	3	4
A	B	C	D

The corresponding control points are 1,2,3 and 4 respectively. There are 16 elemental disturbance velocities to be calculated. From (3.5), (3.7), (3.14) and (3.21), we know the elemental disturbance velocity of u-component includes two parts, one is the symmetric part, $u_s = u_1 + u_2$, and the other is the free wave part, $u_f = u_3$. So in this example, the 16 elemental disturbance velocities of u-components can be written as:

$$u(I,J) = u_s(I,J) + u_f(I,J) \quad I=A, B, C, \text{ and } D, J=1, 2, 3 \text{ and } 4$$

Since u_s is symmetric and the area of each source panel is the same, we have:

$$u_s(A,1) = u_s(B,2) = u_s(C,3) = u_s(D,4)$$

$$u_s(A,2) = u_s(B,3) = u_s(C,4) = u_s(B,1) = u_s(C,2) = u_s(D,3)$$

$$u_s(A,3) = u_s(B,4) = u_s(C,1) = u_s(D,2)$$

$$\text{and } u_s(A,4) = u_s(D,1)$$

Hence, only $u_s(A,4)$, $u_s(B,4)$, $u_s(C,4)$ and $u_s(D,4)$ will be calculated. If the control point is in front of the source panel, the free wave part, u_f , has no contribution. We also have

$$u_f(B,1) = u_f(C,2) = u_f(D,3) = u_f(C,1) = u_f(D,2) = u_f(D,1) = 0$$

and similarly

$$u_f(A,1) = u_f(B,2) = u_f(C,3) = u_f(D,4)$$

$$u_f(A,2) = u_f(B,3) = u_f(C,4)$$

$$u_f(A,3) = u_f(B,4)$$

Only $u_f(A,4)$, $u_f(B,4)$, $u_f(C,4)$ and $u_f(D,4)$ will be calculated.

From (3.6), (3.8) and (3.20), $w_1 + w_2$ has the antisymmetric property, and the same technique can be used to calculate the elemental disturbance velocity of w -component. Thus only the elemental disturbance velocities of each panel with respect to the last control point have to be calculated. In the case of 200 panels with 200 control points, the number of calculations will be reduced from 40,000 to 2,000.

In the second part of the computational scheme, we first find the second derivatives, $\frac{\partial^2 y}{\partial x^2}$ and $\frac{\partial^2 y}{\partial z^2}$, at each intersecting point of waterlines and stations, thus the coefficients of each segment function can be calculated. Then the y-coordinate and the first derivatives, $\frac{\partial y}{\partial x}$ and $\frac{\partial y}{\partial z}$, at any point on the ship hull can be obtained by interpolation. If the ship hull is smooth, we can generate the ship lines very accurately with this method. The number of iterations for finding the "linearized hull" depends on the value of the Froude number and the shape of the ship hull. To save computer time, we can use the "linearized hull" of a lower Froude number as the trial value for a higher Froude number. Generally, the number of iterations is between 40 and 60 for the convergence criterion 1×10^{-4} .

Two models have been selected for computation: Wigley model 3012, a mathematical hull form, and Series 60 block 60, a conventional merchant ship hull. The geometries of the models are given as follows:

(1) Wigley model 3012

$$B/L = 0.1, H/L = 0.0625, C_B = 0.444, C_{PR} = 0.667$$

$$C_x = 0.667, C_s = 0.661 \text{ and } L/L_{pp} = 1.000 \text{ (where } L=LWL)$$

The hull surface is defined by

$$y = \frac{B}{2} [1 - (\frac{2x}{L})^2] [1 - (\frac{z}{H})^2]$$

(2) Series 60 block 60

$$B/L_{PP} = 0.1333, \quad H/L_{PP} = 0.0533, \quad C_B = 0.600, \quad C_{PR} = 0.614$$

$$C_x = 0.977, \quad C_s = 0.710 \quad \text{and} \quad L/L_{PP} = 1.0167 \quad (\text{where } L = \text{LWL})$$

The ship-offsets are in table 1, the bow and stern contours and lines are shown in Fig. 3 and Fig. 4.

The computed results of the wave-making resistance, wave profile, isobars and flow directions along the ship hull are shown in Fig. 5 to Fig. 21. The wave resistance and wave profile curves are plotted over the experimental curves from reference [12] for comparison.

Generally speaking, Guilloton's method gives very good results which are close to the experimental results for Froude number from 0.25 to 0.35 approximately. Especially, in the wave resistance curves, there are no large humps and hollows which usually exist in Michell's resistance curve (see Fig. 5 and Fig. 12). Unfortunately, there are no experimental results to compare with the calculated isobars and flow directions, but from the tendency of the isobars and flow directions, it appears in the correct sense because at the free surface the streamline and isobar coincide, below the free surface, the streamlines have a stronger tendency to go down near the bow and come up near the stern as in [13]. Comparing with series'60 block 60, Wigley model 3012 has smaller "breadth"

and simpler geometrical shape, the variation of its isobars and flow directions are smaller, especially around the stern.

In the last two figures, Fig. 22 and Fig. 23, the calculated wave-making resistance of Wigley hull and Series 60 by Guilloton's method are compared with the results obtained by other researchers. The computational method developed in this thesis has demonstrated its effectiveness and the computed results are quite consistent with that of others, even showing some improvement! The deviations may be due to different numerical techniques for computation, such as the number of source panels, the method of curve fitting, the convergence criterion, etc.

CHAPTER 4

DISCUSSION AND CONCLUDING REMARKS

The main purpose of this thesis is to account for some nonlinear effects of steady ship-flow problem by Guilloton's transformation. Although there is no rigorously theoretical background for Guilloton's method, the computed results have shown that the prediction of the ship flow for a certain Froude number range is very good. In the range of Froude numbers, roughly between 0.25 and 0.35, we may infer that just considering the flow in the vicinity of the ship hull, it could be more important to satisfy the hull and free surface boundary conditions than the field equation. The nonlinear effects included in Guilloton's method are to reduce the oscillatory behaviour of the wave resistance curve based on the thin ship theory and to shift the phase of the wave elevation along the ship hull as that of other higher order theories. [14]. From the definition of the fluid domain in the steady ship-flow problem, Guilloton's method seems more reasonable than the thin ship theory, since the domain is defined below the "disturbed" free surface and out of the interior region of the ship. Another feature of Guilloton's method is to correct the paradox of the thin ship theory which implies that the wave resistance is the same no matter which direction the ship moves, bow or stern ahead. But due to the results of the transformation, the wave-making resistances for these two cases will be different.

and reliable methods to obtain reasonable results for
the stage of preliminary design of conventional ships.

The reasons why the calculated results do not match the experimental results at high Froude number may be due to:

- (a) The sinkage and trim are not considered by Guilloton's method.
- (b) The isobars calculated by equation (4.3) are no longer correct.
- (c) The thin ship theory may not apply to the linearized hull since the distortion of hull geometry is too excessive.
- (d) It can be erroneous for the linearized solution to satisfy the field equation in RS.

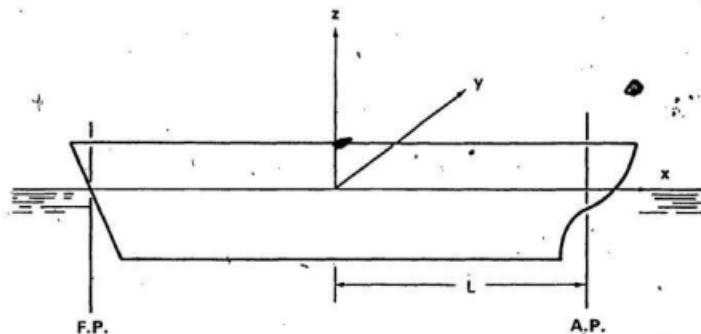
For a ship with a flat bottom, such as S^éries 60 block 60, the bottom part of the hull can not be described by the hull equation, $y = f(x, z)$, so that the calculated flow near the bottom can not exactly correspond to the real flow, even though good results for wave-making resistance are obtained. The calculated flow based on the thin ship theory or Guilloton's method may not be good enough to form the starting point of the boundary layer calculation for ship hulls.

It appears that Guilloton's method for solving the steady ship-flow problem has many disadvantages. The difficulties of the flow problem still can not totally be resolved. However, comparing with other sophisticated methods, for example: higher order theory, finite element method and finite difference method, we may say that Guilloton's method is one of the simplest

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COORDINATE SYSTEM



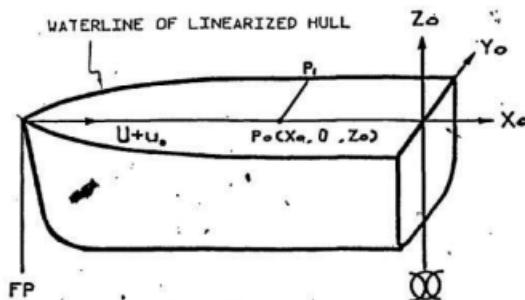
x, y, z Translating coordinate system with x in the opposite direction to the ship's forward motion, z vertically upward, and the origin at the intersection of the planes of the undisturbed free-surface and the midship section.*

x', y', z' Coordinate system fixed in ship and coinciding with the $x-y-z$ system.

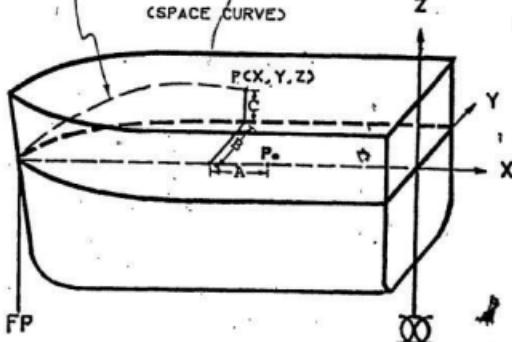
*Midship section is, by definition, at the midpoint between perpendiculars.

FIGURE 1

GEOMETRY OF GUILLOTON'S TRANSFORMATION



WAVE ELEVATION OR ISOBAR ALONG REAL HULL
(SPACE CURVED)



$A(x_0, z_0)$

$B(x_0, z_0)$: Displacements of Guilloton's transformation
 $C(x_0, z_0)$

FIGURE 2

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TABLE I — TABLE OF OFFSETS

SERIES 60, $C_B = 0.60$
(FROM TODD, 1953)

Half breadths of waterline given as fraction of maximum beam on each waterline

Model = 4210W
W.L. 1.00 is the designed load waterline

Forebody prismatic coefficient = 0.581
Afterbody prismatic coefficient = 0.646
Total prismatic coefficient = 0.614

Sta.	Tan.	Waterlines							Area as fraction of max. area to 1.00 W.L.
		0.075	0.25	0.50	0.75	1.00	1.25	1.50	
FP	0.000	0.000	0.000	0.000	0.000	0.000	0.020	0.042	0.000
½	0.009	0.032	0.042	0.041	0.043	0.051	0.076	0.120	0.042
1	0.013	0.064	0.082	0.087	0.090	0.102	0.133	0.198	0.085
1½	0.019	0.095	0.126	0.141	0.148	0.160	0.195	0.278	0.135
2	0.024	0.127	0.178	0.204	0.213	0.228	0.270	0.360	0.192
3	0.055	0.196	0.294	0.346	0.368	0.391	0.440	0.531	0.323
4	0.134	0.314	0.436	0.502	0.535	0.562	0.607	0.683	0.475
5	0.275	0.466	0.589	0.660	0.691	0.718	0.754	0.804	0.630
6	0.469	0.630	0.733	0.802	0.824	0.841	0.862	0.889	0.771
7	0.666	0.779	0.854	0.906	0.917	0.926	0.936	0.946	0.880
8	0.831	0.898	0.935	0.971	0.977	0.979	0.981	0.982	0.955
9	0.945	0.964	0.979	0.996	1.000	1.000	1.000	1.000	0.990
10	1.000	1.000	1.000	1.000	1.000	1.000	1.000	1.000	1.000
11	0.965	0.982	0.990	1.000	1.000	1.000	1.000	1.000	0.995
12	0.882	0.922	0.958	0.994	1.000	1.000	1.000	1.000	0.977
13	0.767	0.826	0.892	0.962	0.987	0.994	0.997	-1.000	0.938
14	0.622	0.701	0.781	0.884	0.943	0.975	0.990	0.999	0.863
15	0.463	0.560	0.639	0.754	0.857	0.937	0.977	0.994	0.750
16	0.309	0.413	0.483	0.592	0.728	0.857	0.933	0.975	0.609
17	0.168	0.267	0.330	0.413	0.541	0.725	0.844	0.924	0.445
18	0.065	0.152	0.193	0.236	0.321	0.536	0.708	0.834	0.268
18½	0.032	0.102	0.130	0.156	0.216	0.425	0.626	0.769	0.187
19	0.014	0.058	0.076	0.085	0.116	0.308	0.530	0.686	0.109
19½	0.010	0.020	0.020	0.022	0.033	0.193	0.418	0.579	0.040
AP	0.000	0.000	0.000	0.000	0.000	0.082	0.270	0.420	0.004
Max half beam*	0.710	0.866	0.985	1.000	1.000	1.000	1.000	1.000	

*As fraction of maximum load waterline beam.

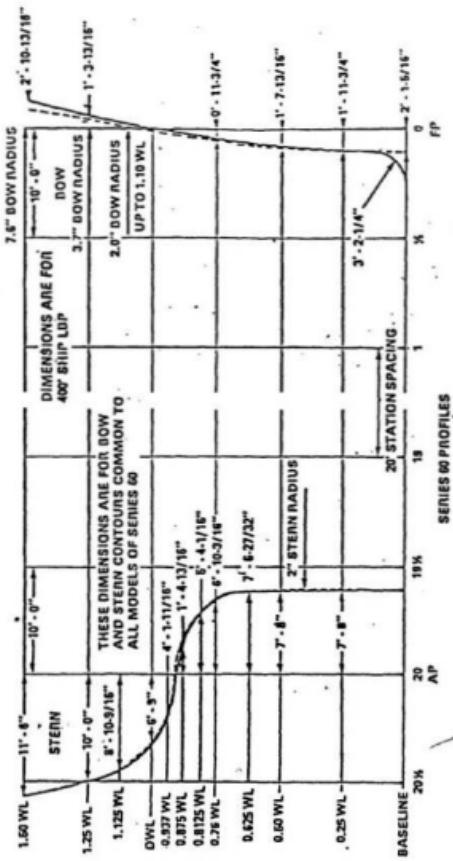
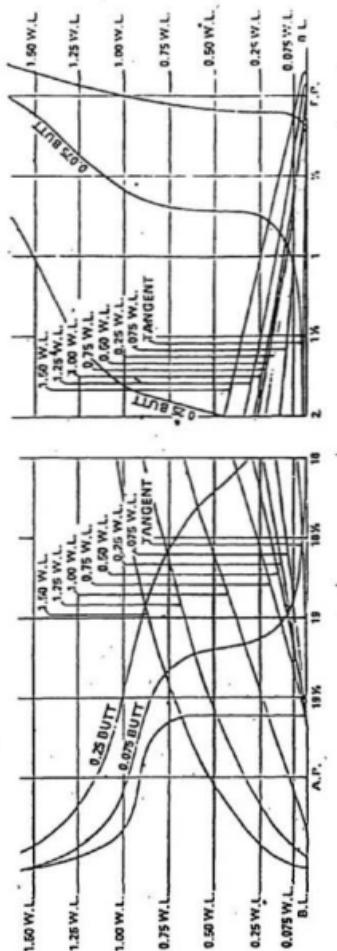
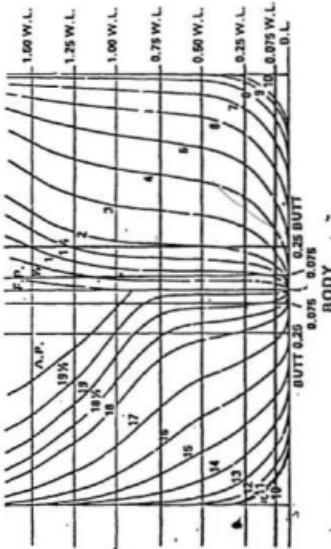


FIGURE 3—How and Stern Contours
(from Todd, 1963)



BOW

STERN



BODY

FIGURE 4—Lines of Series 60, $C_0 = 0.60$ Model 4210V
(from Todd, 1953)

WIGLEY HULL - RESISTANCE COEFFICIENT

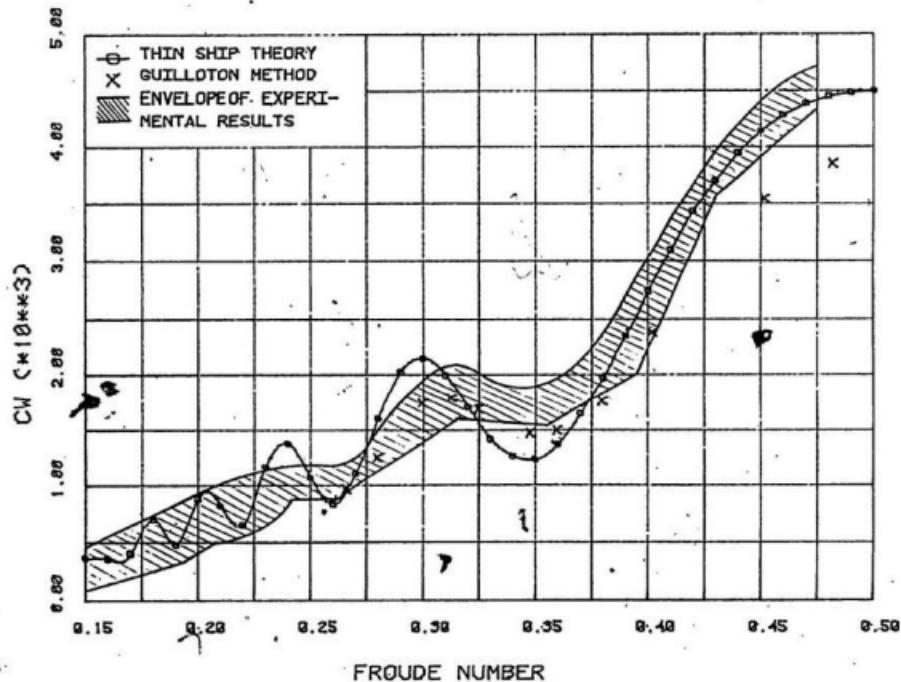


FIGURE 5

WIGLEY HULL - WAVE PROFILE, FOR $FN=0.266$

— EXPERIMENTAL RESULT
 X THEN SHIP THEORY
 ▲ GUILLOTON METHOD

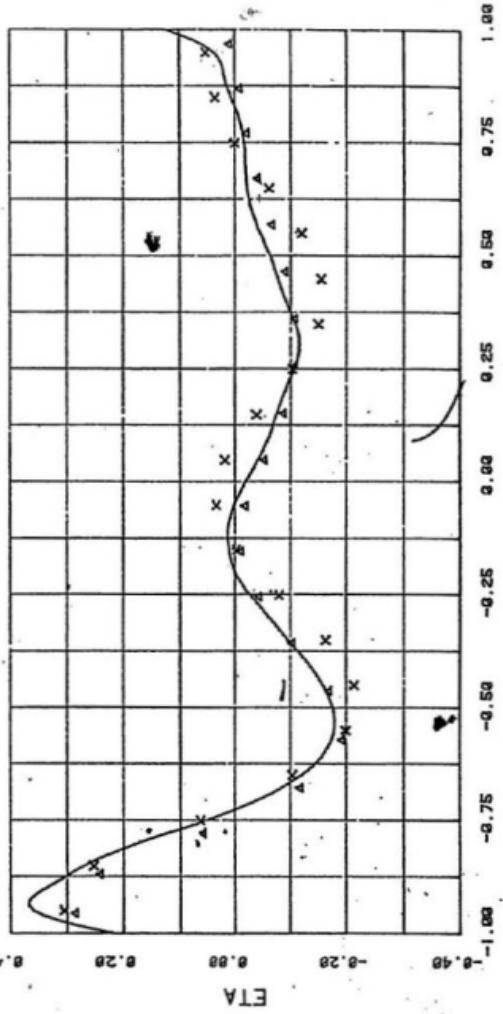


FIGURE 6

WIGLEY HULL - ISOBARS FOR FN=0.266

56

WAVE PROFILE

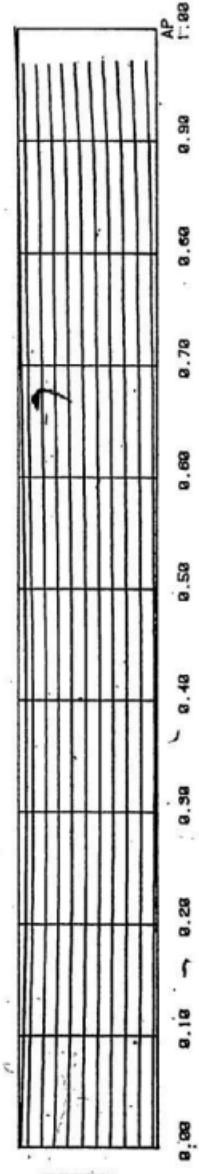
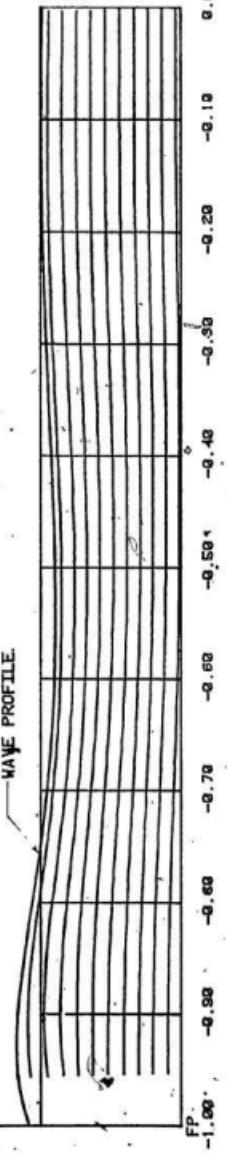
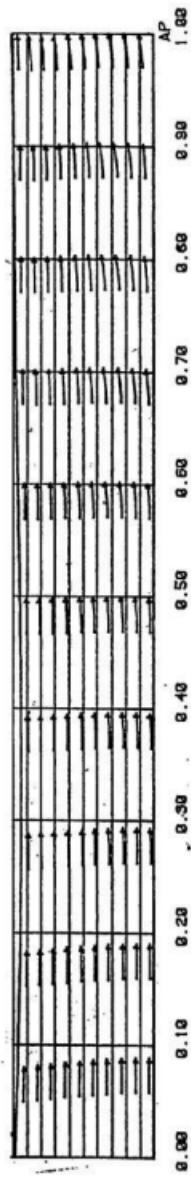
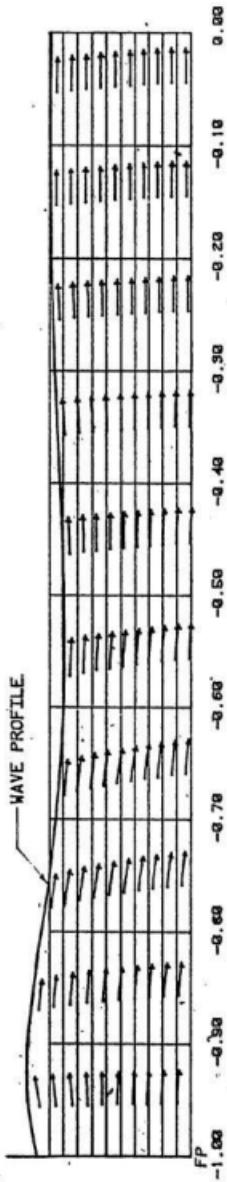


FIGURE 7

WIGLEY HULL - FLOW DIRECTIONS FOR. FN=0.266

57



WIGLEY HULL - WAVE PROFILE FOR $F_N=0.348$

EXPERIMENTAL RESULT

X THIN SHIP THEORY

A SUDLOVTON METHOD

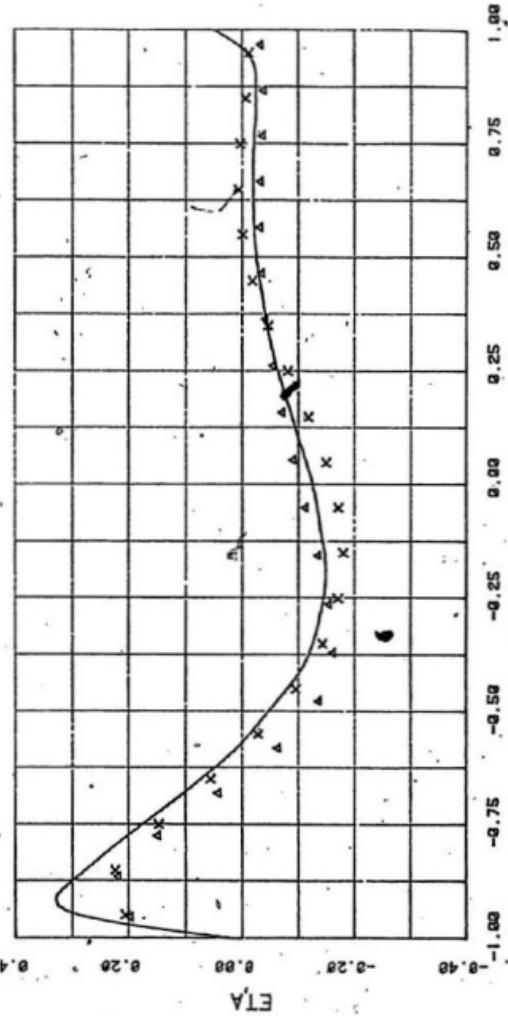
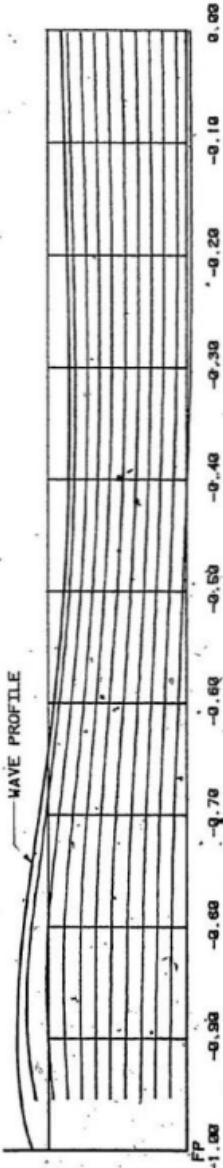


FIGURE 9

WIGLEY HULL - ISOBARS FOR FN=0.348

59

WAVE PROFILE



-1.00 -0.50 -0.30 -0.20 -0.10 -0.05 0.00 0.10 0.20 0.30 0.40 0.50 0.50

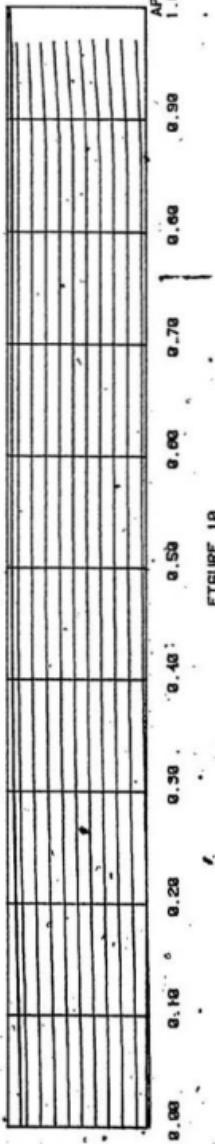


FIGURE 10

WIGLEY HULL - FLOW DIRECTIONS FOR FN=0 : 348

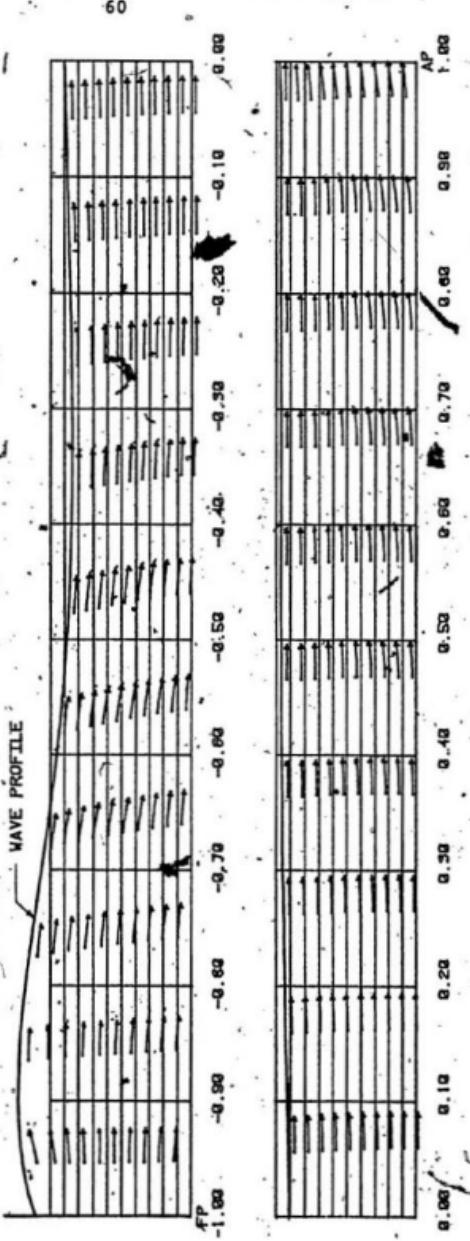


FIGURE 10

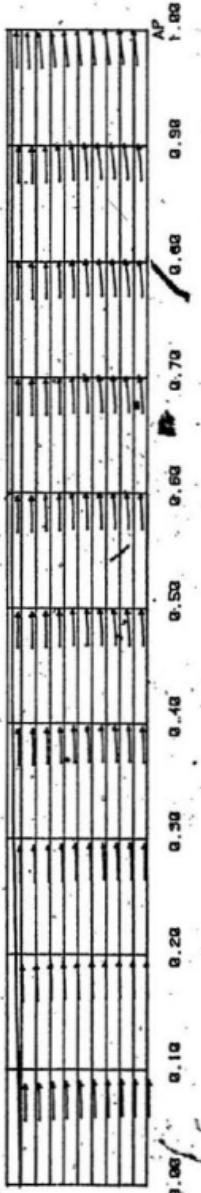
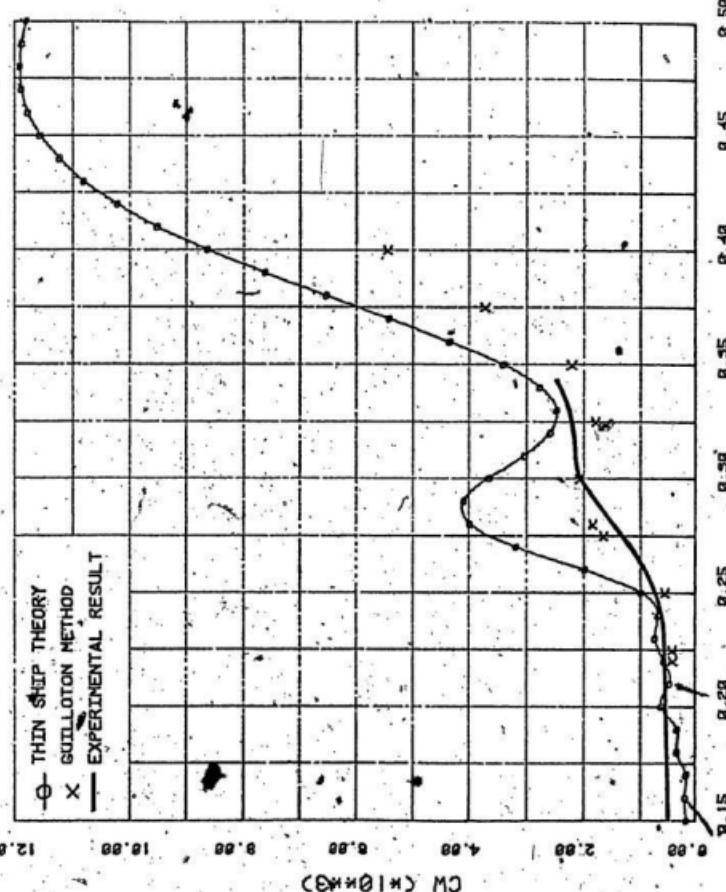


FIGURE 11

SERIES 60 BLOCK 60

- RESISTANCE COEFFICIENT

FIGURE 12
CM CH10H43

SERIES 60 BLOCK 60 - WAVE PROFILE FOR FN=0.220

— EXPERIMENTAL RESULT

X THIN SHIP THEORY

▲ SUBLIMATION METHOD

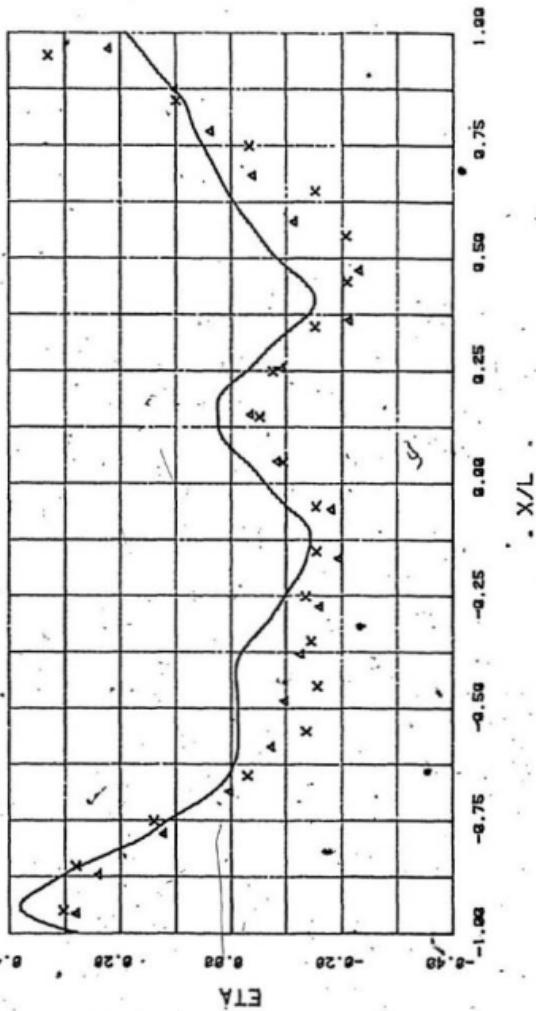
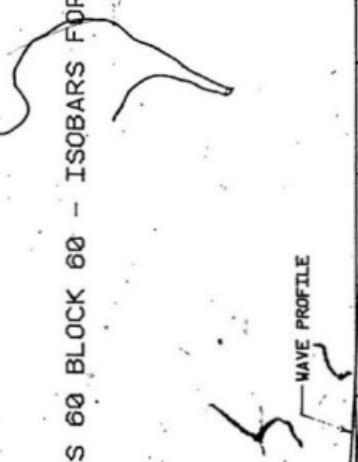


FIGURE 13

SERIES 60 BLOCK 60 - ISOBARS FOR FN=0.220



WAVE PROFILE

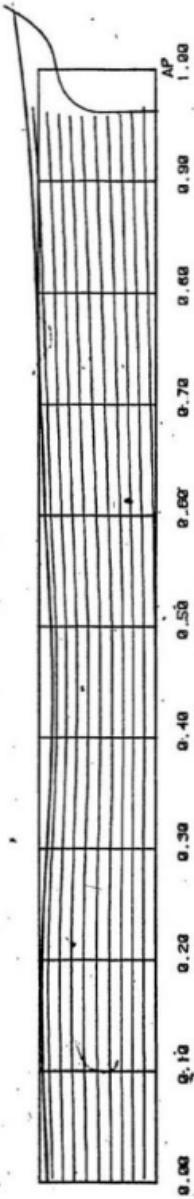
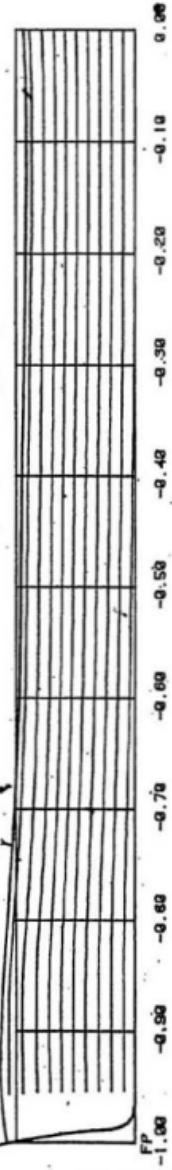
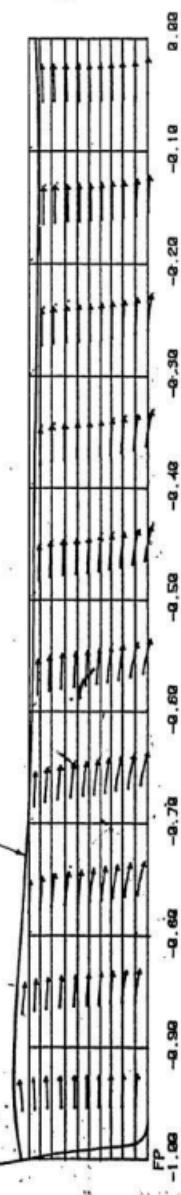


FIGURE 14

SERIES 60 BLOCK 60 - FLOW DIRECTIONS FOR FN=0.220

64

WAVE PROFILE



-1.00 -0.90 -0.80 -0.70 -0.60 -0.50 -0.40 -0.30 -0.20 -0.10 0.00

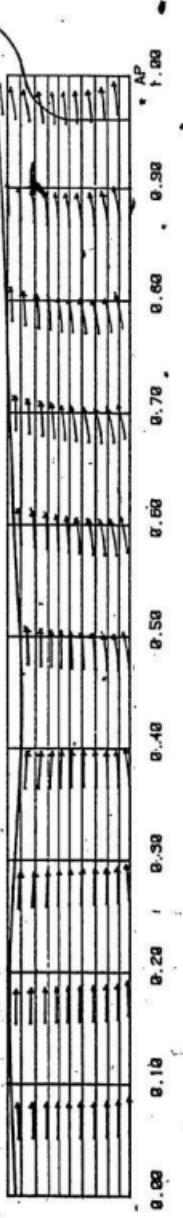


FIGURE 16

SERIES 60 BLOCK 60 - WAVE PROFILE FOR FN=0.280

EXPERIMENTAL RESULT
 X THIN SHIP THEORY
 ▲ SQUILLON METHOD

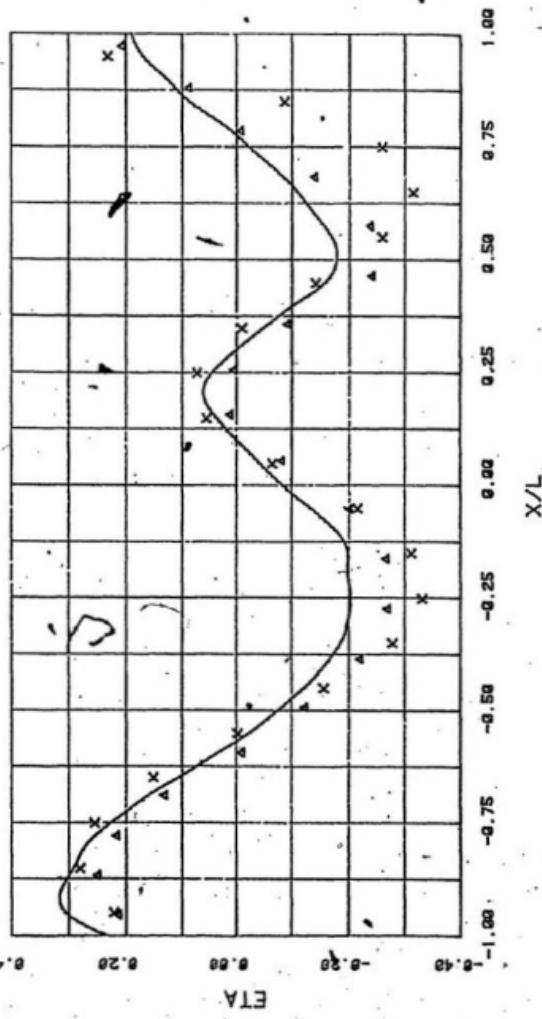


FIGURE 16

SERIES 60 BLOCK 60 - ISOBARS FOR FN=0.280

66

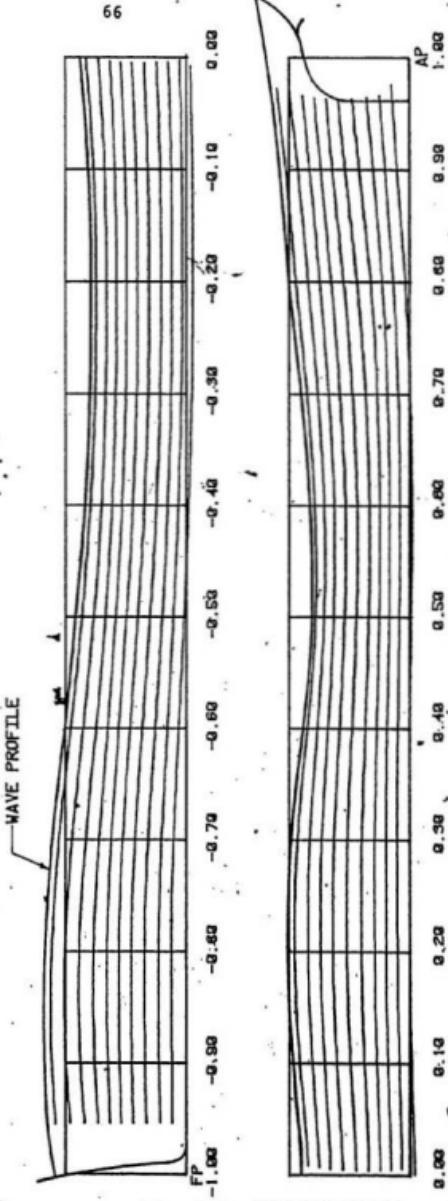


FIGURE 16

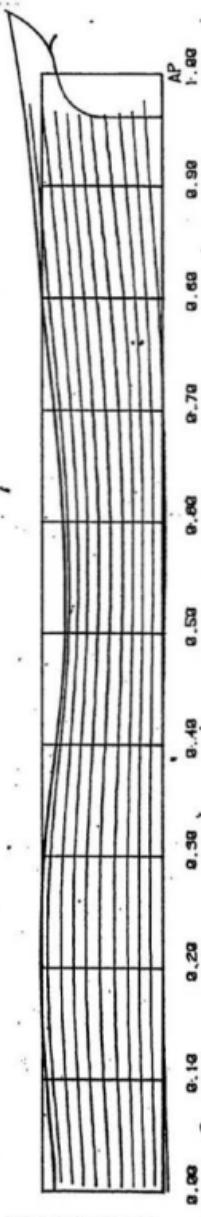


FIGURE 17

SERIES 60 BLOCK 60 - FLOW DIRECTIONS FOR FN=0.280

WAVE PROFILE

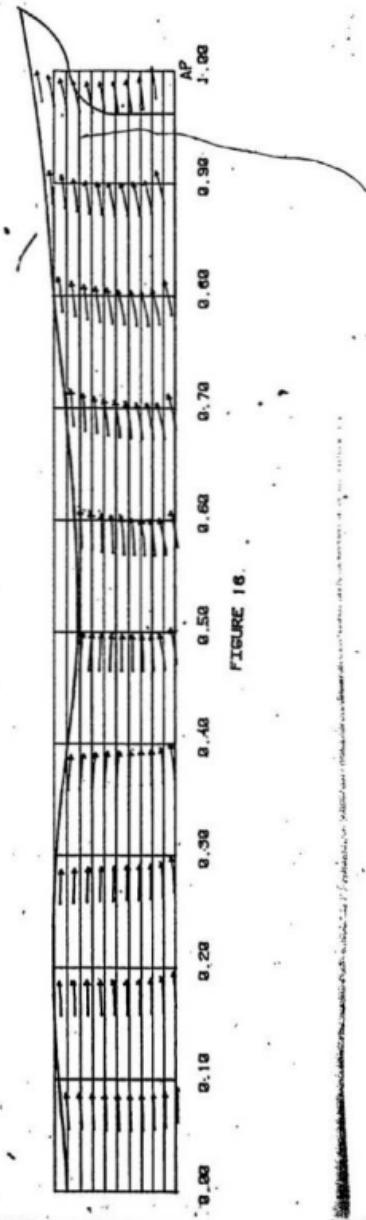
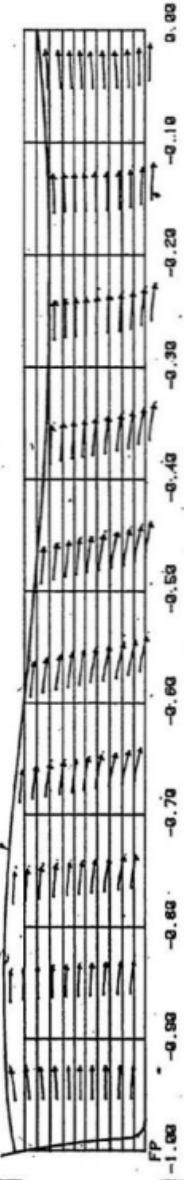


FIGURE 18.

SERIES 60 BLOCK 60 - WAVE PROFILE FOR $FN=0.35Q$

— EXPERIMENTAL RESULT

X THIN SHIP THEORY

▲ GUILLOTTON METHOD

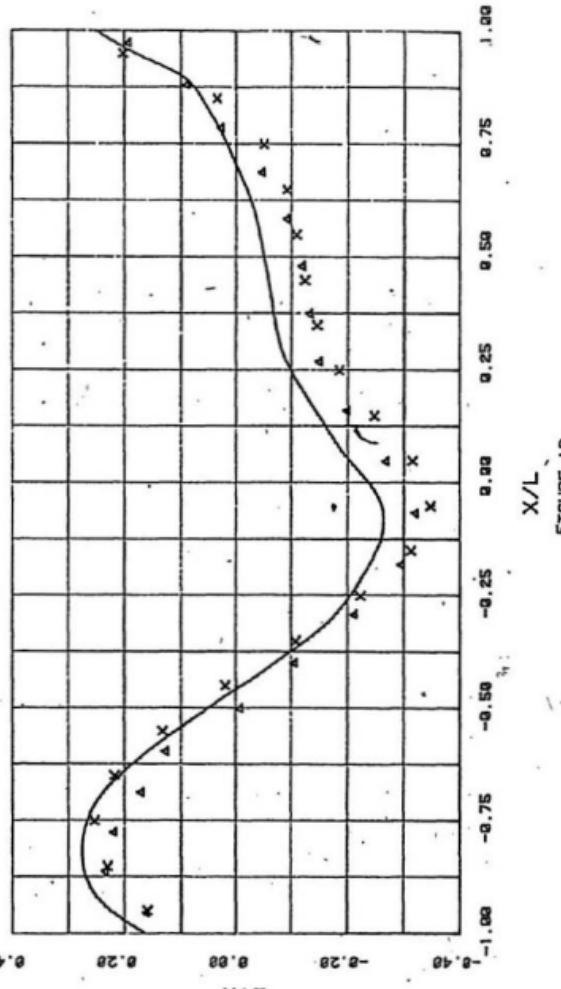


FIGURE 10

SERIES 60 BLOCK 60 - ISOBARS FOR FN=0.350

69

WAVE PROFILE

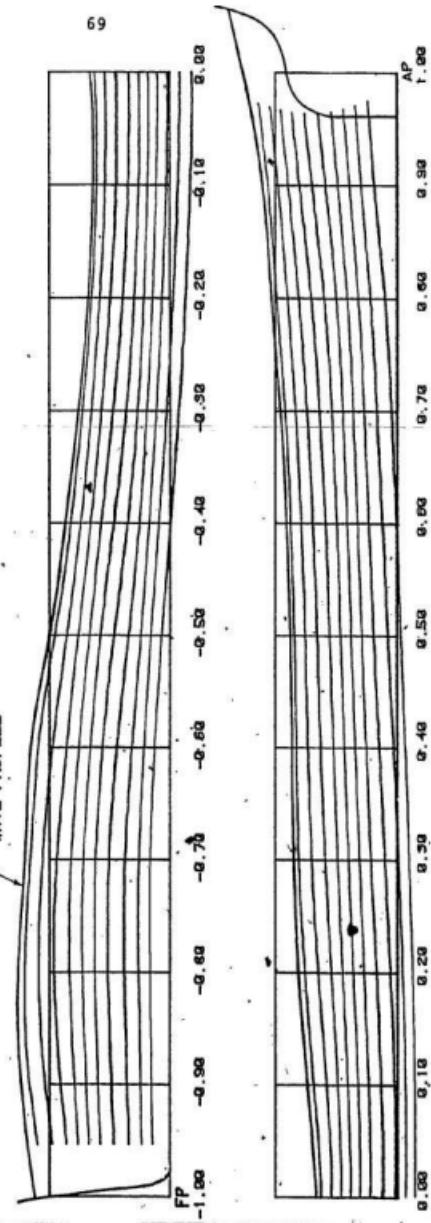


FIGURE 28

SERIES 60 BLOCK 60 - FLOW DIRECTIONS FOR FN=0, 350

70

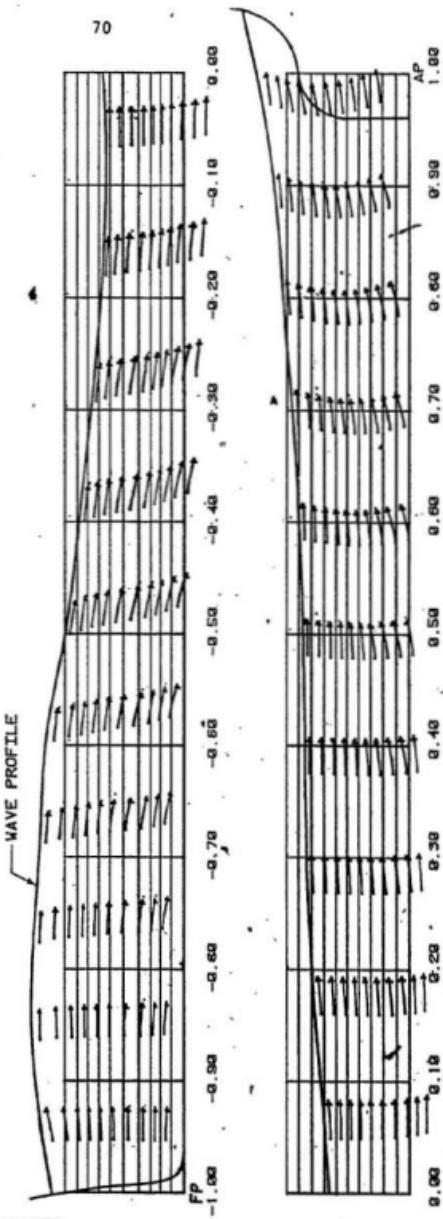
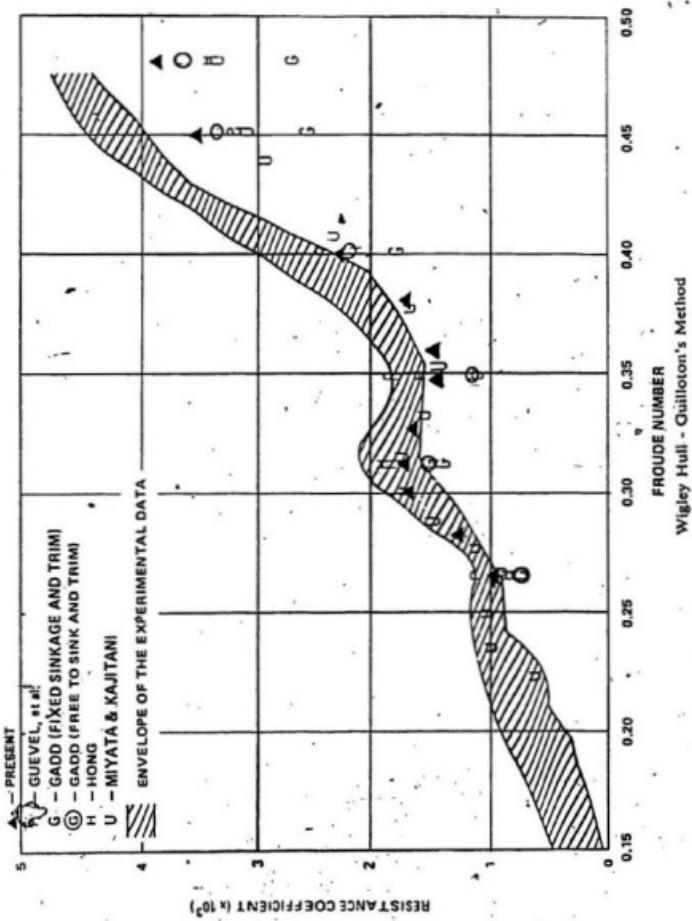
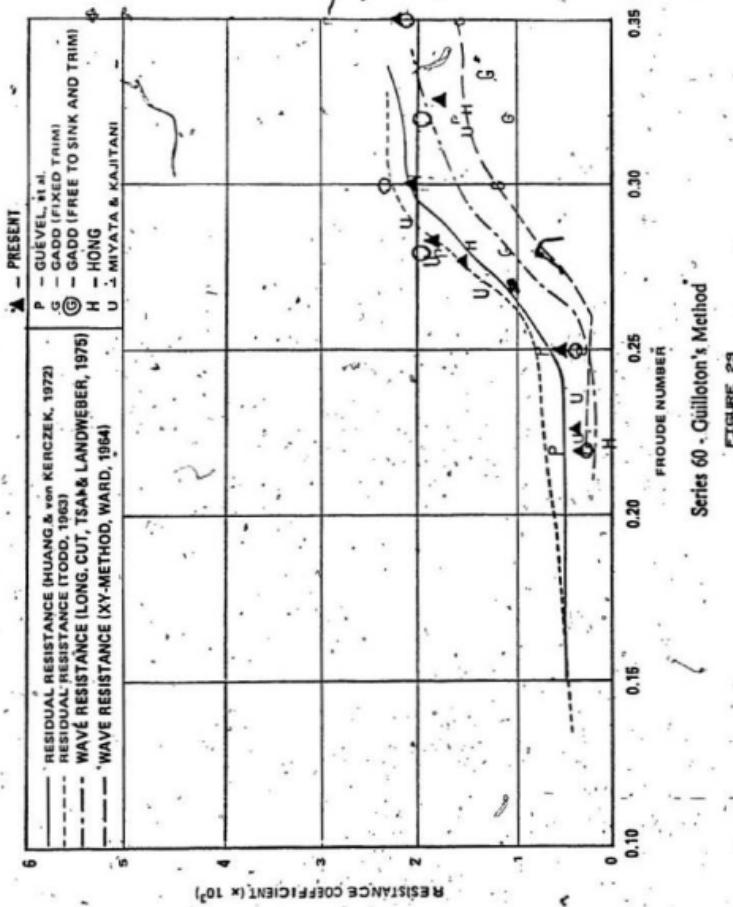


FIGURE 21





Series 60 - Quilliotin's Method

FIGURE 29

APPENDIX : COMPUTER PROGRAM

```

C-----TITLE OF THE THESIS-----
C-----ANALYSIS OF SHIP FLOW IN AN IDEAL FLUID USING GUILLOTON'S
C-----METHOD AND SPLINE FUNCTIONS
C-----FUNCTION: THIS PROGRAM COMPUTES THE WAVE RESISTANCE COEFF. AND
C-----WAVE PROFILES FROM THE GIVEN SHIP-OFFSETS AND THE FROUDE
C-----NUMBERS BY THIN SHIP THEORY AND GUILLOTON'S METHOD.
C-----WAVE RESISTANCE COEFFICIENT IS      CW=RU/(8.54*RH0*(U*U*6))
C-----NONDIMENSIONAL WAVE ELEVATION IS     ETA=Z/(U/(2*G))
C-----DIMENSION ETAC(28),ETA1(28),FDL(14),SMC(28,18),SM1(28,18)
COMMON /A/ NX,NX1,NX2,NZ,NZ1,NZ2,XL,T,XC21,ZC11,XCC(28),ZCC(18)
1,XB,WB,XS,XE
COMMON /SA/ NFN,FNC(28)
COMMON /SC/ DFY(22,28),XA(22),ZA(28),YX2(22,28),YZ2(22,28),
COMMON /SD/ SHCP(28),XL,B,TD,WB1
COMMON /S/ SC(28,18),SI(28,18),S2(28,18),UI(28,18)
NX=21
NX1=NX-1
NZ=11
NZ1=NZ-1
XL=2.8
TYPE *,'ENTER OUTPUT DATAFILE NAME'
ACCEPT 1,FIL1
FORMAT(44A4)
CALL ASSIGN(2,FIL1)
CALL INPUT
CALL POINT
C TYPE *,'POINT FINISHED'
CALL CURFIT(6D)
C-----OUTPUT: SHP,SMCI,J0
100 WRITE(2,100) SHP
FORMAT(1X,28A4D)
WRITE(2,2) WRITE(2,2) 'L=' ,XL, ' T=' ,TD, ' B=' ,B, ' WB=' ,WB1
WRITE(2,2) ' SOURCE STRENGTH BY THIN SHIP THEORY'
WRITE(2,2)
DO 5 I=1,NX1
  WRITE(2,2) SMCI,I,J0,J1,180
CONTINUE
C-----NFM=1
C-----FNC1=0.228
DO 10 I=1,NFN
  FR=FNC(I)
  XXB=1./2.*FR*FR*TD
  DO 11 J=1,NX1
    DO 28 JU=1,NZ1
      SCII,JU=SMCII,JU
      SI(XI,JU)=SMCII,JU
      S2(XI,JU)=SMCII,JU
28 CONTINUE
  CALL BULTON(CB1)
  CALL KICHSM(CV)
  CALL KICHSM1(CV1)
  CALL WAVECSM(SH1,ETA,ETA1)

```

```

C-----  

C      OUTPUT: FR, SH1, CH1, CH1, ETA, ETA1  

      WRITE(2,48)  

      WRITE(2,49) 'SOURCE STRENGTH BY GUILLOTON METHOD FOR FNH',FR  

      WRITE(2,50)  

      DO 50 II=1,NX1  

        WRITE(2,51) CSH1(II),JJ,JJ=1,NZ1  

      CONTINUE  

      WRITE(2,52)  

      WRITE(2,53) 'FR=',FR,' CH1=',CH1,' CH1= ',CH1  

      WRITE(2,54)  

      WRITE(2,55) 'CH1 : WAVE RESISTANCE COEFF. BY THIN SHIP THEORY'  

      WRITE(2,56) 'CH1 : WAVE RESISTANCE COEFF. BY GUILLOTON METHOD'  

      WRITE(2,483)  

      FORMAT(1H1)  

      WRITE(2,57) 'WAVE PROFILE FOR FNH',FR  

      WRITE(2,58)  

      DO 58 J=1,NX1  

        WRITE(2,59) 'XL= ',XCC(J), ' ETA= ',ETAC(J), ' ETA1= ',ETA1C(J)  

      CONTINUE  

      WRITE(2,60)  

      WRITE(2,61) 'ETA : WAVE PROFILE BY THIN SHIP THEORY'  

      WRITE(2,62) 'ETA1 : WAVE PROFILE BY GUILLOTON METHOD'  

C-----  

1000  CONTINUE  

10000 STOP  

END  

SUBROUTINE POINT  

COMMON //> NX,NX1,NX2,NZ,NZ1,NZ2,XL,T,X(21),Z(11),XCC20,ZC(18)  

1,XOB,W8,X5,XE  

SX-XL/CN2-1  

SX-T/CN2-1  

DO 18 I=1,NX  

  XCID=1.+SX*(CI-1)  

  IF(CI.EQ.ED).NOT.GO TO 18  

  XCID=1.+SX*(CI-0.5)  

18  CONTINUE  

DO 28 I=1,NZ  

  ZCID=T+SZ*(CI-1)  

  IF(CI.EQ.ED).NOT.GO TO 28  

  ZCID=T+SZ*(CI-0.5)  

28  CONTINUE  

RETURN  

END  

SUBROUTINE INPUT  

C*****  

C      FUNCTION: THIS SUBROUTINE READS INPUT DATA FROM INPUT DATAFILE  

C              AND NORMALIZES THE SHIP-OFFSETS.  

C              THE NONDIMENSIONAL LBF IS 2.    (FROM -1. TO 1.)  

C*****  

DIMENSION FILE(44)  

COMMON //> NX,NX1,NX2,NZ,NZ1,NZ2,XL,T,X(21),Z(11),XCC20,ZC(18)  

1,XOB,W8,X5,XE  

COMMON //> FNH,FNC20  

COMMON //> MFT,DFX200,DFZ200,DF{20,20},NOFC20  

COMMON //> SHDFT200,XLL,B,TD,W81  

TYPE *, 'ENTER OFFSET DATAFILE'  

ACCEPT 1,FILE  

FORMAT(4444)  

CALL ABEND1,FILE1  

READ(1,2000) SHDFT  

FORMAT(C80A44)

```

```

READ(1,10) NPN
10 READ(1,10) (FNCLD,I=1,NPN)
READ(1,10) XLL,B,TD,WS1
SCALE=XLL/2.
WS=WS1/(SCALE*SCALE)
T=TD*2./XLL
READ(1,10) NST
DO 10 I=1,NST
  READ(1,10) OFXCI,NOFCI
  NI=NOFCI
  DO 20 J=1,NI
    READ(1,10) OFZCI,J,D,OFYCI,J
    OFZCI,J=OFZCI,J-TD*2./XLL
    OFYCI,J=OFYCI,J*2./XLL
20  CONTINUE
10  CONTINUE
CALL CLOSE(1)
RETURN
END
SUBROUTINE CURFIT(SM)
DIMENSION SM(28,180)
DIMENSION FILE(4),XX(26),YY(26),Y2(26)
DIMENSION YXX(25),YX(22,28),OFY(26,28)
COMMON /A/ NX,NX1,NX2,NZ,NZ1,NZ2,XL,T,X(21),Z(11),XC(28),ZC(28),
1,XX9
COMMON /Y/ NST,OX(26),OZ(26,28),OY(26,28),NC(26),CC(26)
COMMON /Y/ OFYY(22,28),XA(22),ZA(28),YX(22,28),YZ(22,28)
NDC2=NX+1
NZ2=NZ+9
NDC1=NZ2
NZ2=NZ2
XL1=XL
T1=T
CALL PNTCOL1,T1,NDC1,NZ2
DO 30 I=1,NST
  NI=NCI
  DO 40 J=1,NI
    XX(J)=OZCI,J
    YY(J)=OFYCI,J
40  CONTINUE
CALL CUBIC1(NI,XX,YY,Y2)
ID=1
CALL INTRPC1,NZ2,ZA,NI,XX,YY,Y2,YOO
DO 50 J=1,NZ2
  OFYCI,J=YOO(J)
50  CONTINUE
CONTINUE
DO 60 J=1,NZ2
  DO 70 I=1,NST
    YYCI=OFYCI,J
70  CONTINUE
CALL CUBIC1(NST,OX,YY,Y2)
ID=1
CALL INTRPC1,NDC1,XA,NST,OX,YY,Y2,YOO
DO 80 I=1,NDC1
  OFYYCI,J=YOO(J)
80  CONTINUE
CONTINUE
DO 90 I=1,NDC1
  DO 100 J=1,NZ2
    XXGJ=OZCI,J
    YYGJ=OFYGYJ,J
100 CONTINUE

```

```

188 CONTINUE
CALL CUBIC1(N02,XX,YY,Y2)
ID=2
CALL INTRP2D,N02,XX,N02,YY,Y2,Y00
DO 118 J=1,N02
Y(XJ,I)=Y00(J)
CONTINUE
118 CONTINUE
II=8
DO 128 I=1,N02
IFCI.EQ.13 GO TO 128
IFCI.EQ.N02) GO TO 128
II=II+1
JJ=8
DO 138 J=1,N02
IFCJ.EQ.1 AND .LE.6) GO TO 138
IFCJ.EQ.7) GO TO 138
IFCJ.EQ.N02-3) GO TO 138
JJ=JJ+1
SH(II,J)=2.*Y(X,I,J)
138 CONTINUE
128 CONTINUE
DO 148 J=1,N02
DO 168 I=1,N02
XX(I)=X(A(I))
YY(I)=OPYY(I,J)
168 CONTINUE
CALL CUBIC1(N02,XX,YY,Y2)
DO 169 I=1,N02
Y2(I,J)=Y2(I)
169 CONTINUE
148 CONTINUE
DO 178 I=1,N02
DO 188 J=1,N02
XX(J)=Z(A(J))
YY(J)=OPYY(J,J)
188 CONTINUE
CALL CUBIC1(N02,XX,YY,Y2)
DO 198 J=1,N02
Y2(J,J)=Y2(J)
198 CONTINUE
178 CONTINUE
RETURN
END
SUBROUTINE INTRP2D,N2,ZA,NI,XX,YY,Y2,OUT)
DIMENSION ZA(N2),XX(NI),YY(NI),Y2(NI),OUT(N2)
DIMENSION HC(28),HY(28),AC(28),BC(28),CC(28),DC(28)
DO 98 II=1,NI-1
HC(II)=4*(XX(II)+1)-XX(II)
HY(II)=YY(II)+1)-YY(II)
98 CONTINUE
DO 78 II=1,NI-1
AC(II)=(Y2(II)+1)-Y2(II))/(8.*HY(II))
BC(II)=8.*HY(II)
CC(II)=HY(II)/HC(II)-HC(II)=(Y2(II)+1)*2.*Y2(II))/8.
DC(II)=YY(II)
78 CONTINUE
DO 18 I=1,N2
ZZ=Z(A(I))
DO 28 J=1,NI-1
IFCZ.EQ.10(J).AND.ZZ.LE.X(X(J)+1)) GO TO 28
CONTINUE

```

```

38      DEL=Z2-Z(XCJ)
      IF(CID,ED,1) OUT(CJ)=CC(A(J))DEL+BC(J)DEL+CC(J)DEL+DC(J)
      IF(CID,ED,2) OUT(CJ)=CC3.PAG(J)DEL+2.MB(J)DEL+CC(J)
18      CONTINUE
      RETURN
      END
      SUBROUTINE PNTOL(T,N02,N22)
      COMMON /SC/ OFYY(22,28),XC(22),ZC(28),Y2(22,28),Y2Z(22,28)
      SX=XL/XN2-23
      SZ=Y/T*(N22-18)
      DO 18 I=1,N02
         J=I-1
         IFCI,ED,1) XCJ=-1.0
         IFCI,ED,N02) XCJ=1.0
         IFCI,NE,1.AND.I,NE,N02) XCJ=-1.+8X*GJ-0.5
18      CONTINUE
         ZC1=T
         ZC2>ZC1>-62/16.
         ZC3>ZC2>-62/16.
         ZC4>ZC3>-62/8.
         ZC5>ZC4>-62/8.
         ZC6>ZC5>-62/8.
         ZC7>ZC6>-62/2.
         ZC8>ZC7>-62/2.
         DO 28 I=0,N22
            ZCJ>ZC1-1>+62
28      CONTINUE
      RETURN
      END
      SUBROUTINE BULLTON(GH1)
C***** FUNCTION: THIS SUBROUTINE COMPUTES NEW SOURCE STRENGTH BY
C***** BULLTON'S METHOD.
C*****
      DIMENSION FILE(4),SH1C28,183
      COMMON //A/ NX,NX1,NX2,NZ,NZ1,NZ2,XL,T,XC(21),ZC(28),ZC(183
      1,XK8,W,X8,XE
      COMMON /C/ UTC28,18,180,UL(28,18,180)
      COMMON /B/ SC(28,180),S1C(28,180),S2C(28,180),UI(28,180)
      COMMON /SC/ OFY(22,28),XA(22),ZA(28),Y2(22,28),Y2Z(22,28)
      XS=-1.0
      XE=1.0
      CALL INDU2
      ID=1
      29      TYPE *, 'ITERATION',ID
      CALL CHAN0
      IR=0
      CALL TRANCIR,ID)
      IFCL,NE,1) GO TO 38
      IFCD,ED,180) GO TO 38
      ID=ID+1
      30      TO 28
      DO 48 I=1,NX1
         DO 48 J=1,NZ1
            SH1CJ,J=SH1CJ,J
48      CONTINUE
      RETURN
      END
      SUBROUTINE CHAN0
      DIMENSION UC(28,180)
      COMMON //A/ NX,NX1,NX2,NZ,NZ1,NZ2,XL,T,XC(21),ZC(183,XC(28),ZC(183
      1,XK8,W,X8,XE

```

```

COMMON //C/ UTC28,18,180,ULC28,18,180
COMMON //S/ S28,180,S128,180,S2(28,180),UIC28,180
DO 28 I=1,NZ1
DO 28 J=1,NX1
  UIC(J,I)=0.8
  NO=CJ-1>NO(J)+J
  DO 38 ZZ=1,NX1
    IS1=NO(J)-ZZ
    IS2=NO(J)+ZZ
    DO 38 JJ=1,NZ1
      NO(J)=JJ-1>NO(J)+ZZ
      IF(CJ.LE.JD UC(J,J)=UTC281,JJ,JD
      IF(CJ.GT.JD UC(J,J)=ULC282,JJ,JD
  CONTINUE
  DO 48 JJ=1,NX1
  DO 48 J=1,NZ1
    UIC(J,J)=UIC(J,J)+UC(J,J)+S1C(J,J)
  CONTINUE
  RETURN
END

SUBROUTINE TRANCER,JD
DIMENSION YC280,XH(21),YH(21),ZH(21),UP(28),TH(28),TX(28)
DIMENSION Y21(28),XMC(28),180,ZMH(28,180)
COMMON //A/ MX,NX1,NX2,NZ,NZ1,NZ2,XL,T,XC(21),ZC(28),ZC(180)
1,XC8,18,XB,XE
COMMON //S/ S28,180,S128,180,S2(28,180),UIC28,180
F1CA1,C12=C1+AI2/8QRT(C1+C1)=C12
ENAM=0.8
DO 18 ZZ=1,NZ1
  DO 28 JJ=1,NX1
    AI=MUC(J,J)
    CI=S1C(J,J)/2.
    TN(J,J)=F1CA1,C12
    YG(J,J)=TN(J,J)
  CONTINUE
  ZX=1
  CALL XMCAL(CDX,Y,XH)
  DO 38 JJ=1,NX1
    ZH(J,J)=0.01(JD-1.
    ZMH(J,J)=0.01(JD)
  CONTINUE
  DO 48 JJ=1,NX1
    YG(J,J)=CJ,J/2.
    ZH(J,J)=ZC(J,J)-UP(J,J)/280
    ZMH(J,J)=ZH(J,J)
  CONTINUE
  CALL XMCAL(CDX,Y,UP)
  CALL PARTL(CYX1,YZ1,XH,ZH)
  DO 58 JJ=1,NX1
    YG(J,J)=UIC(J,J)
  CONTINUE
  ZX=0
  CALL XMCAL(CDX,Y,UP)
  DO 68 JJ=1,NX1
    YG(J,J)=PCTH(J,D)YX1(J,J)-UP(J,J)*YZ1(J,J)/280
    S1C(J,J)=YG(J,J)
  CONTINUE
  DO 78 JJ=1,NX1
    YG(J,J)=S2(J,J)
  CONTINUE
  DO 88 JJ=1,NX1

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CX=ABS(S2CJ,JD)/S1CJ,JD-1.0
IF(CX.GT.EMAX) EMAX=CX
IF(CX.GT.1.E-40) IR=1
S2CJ,JD=S1CJ,JD
CONTINUE
100 TYPE N, 'ERRMAX', EMAX
IF(CIR.EQ.1) GO TO 100
WRITEC2,WD
WRITEC2,WD 'ISOBARS'
WRITEC2,WD ''
DO 110 J=1,NZ1
WRITEC2,WD ''
WRITEC2,WD 'No. ',J
WRITEC2,WD ''
WRITEC2,WD '#X-COOR.'
WRITEC2,WD 'C0M(J,JD,J=1,NX1)
WRITEC2,WD ''
WRITEC2,WD '#Z-COOR.'
WRITEC2,WD '#ZNM(J,JD,J=1,NX1)
CONTINUE
100 RETURN
END
SUBROUTINE PARTLC(Y1,Y2,XN,ZH)
DIMENSION Y1(220),Y2(220),YT(220),YT2(220)
DIMENSION XN(210),ZH(210)
COMMON /A/ NX,NX1,NX2,NZ,NZ1,NZ2,Y,T,X(21),Z(11),XC(220),ZC(180)
1,XX8,XB,XB,XE
COMMON /SC/ OFYC(22,280),XC(220),ZZ(280),YX2(22,280),Y2(22,280)
DO 30 JD=1,NX1
30 XI=00CJD
ZI=20CJD
DO 40 J=1,NX2-1
40 IFOCI.BE.XC(JD).AND.XI.LE.30CJD+100 GO TO 50
CONTINUE
TYPE N, 'OUT OF RANGE JD=1,XI
IFOCI.BT.XC(NX2)) J=NX2-1
IFOCI.BT.20CJD DELX=00CJD-10CJD-1
IFOCI.LT.XC(1)) J=1
IFOCI.LT.20CJD DELX=00.8
Y2CJD=1.E-10
50 TO 125
DELX=XI-10CJD
125=J
125=J+1
H1=00CX(JD)-10CX(1)
DO 90 J=1,NZ2
90 Y2=OFYC(JD,J)
Y1=OFYC(1,J)
Y22=YX2CJD,J
Y12=YX2CJD,J
Y12=YX2CJD,J
HW=Y2-Y1
A=(Y22-Y12)/(0.1,NH1)
B=0.5AY12
C=H1/H1-H1*(Y22+2.5Y12)/6.
D=V1
YT(JD)=CC(A+DELX)+B+DELX+C+DELX+D
CONTINUE
CALL CUBIC1(XN2,ZZ,YT,YT2)
100 TO 100 J=1,NZ2-1
IFOCI.BE.ZZ(JD).AND.ZZ.LE.ZZ(JD+100) GO TO 110
100 CONTINUE

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IF(ZZ1.GT.ZZ(NZ2)) J=NZ2-1
IF(ZZ1.GT.ZZ(NZ2)) DELZ=ZZ(NZ2)-ZZ(NZ2-1)
IF(ZZ1.LT.ZZ(1)) Y1(CD)=1.E-18
IF(ZZ1.LT.ZZ(1)) 80 TO 126
DELZ=ZZ(1)
IB=J
IE=J+1
H=ZZ(CD)-ZZ(CB)
HY=Y(T2(CD))-Y(T2(CB))
A=(YT2(CD)-YT2(CB))/(C.HD)
B=0.5*YT2(CB)
D=HY/H+HY(YT2(CD)+2.*YT2(CB))/6.
DY=YT2(CB)
Y1(CD)=CC(MA)+DELZ+2.*B+D=DELZ+C
DO 148 J=1,NZ2-1
  IF(ZZ1.GE.ZZ(J).AND.ZZ1.LE.ZZ(J+1)) 80 TO 168
CONTINUE
IF(ZZ1.GT.ZZ(NZ2)) J=NZ2-1
IF(ZZ1.GT.ZZ(NZ2)) DELZ=ZZ(NZ2)-ZZ(NZ2-1)
IF(ZZ1.LT.ZZ(1)) Y1(CD)=1.E-18
IF(ZZ1.LT.ZZ(1)) 80 TO 38
DELZ=ZZ(1)
IB=J
IE=J+1
H=ZZ(CD)-ZZ(CB)
DO 168 J=1,NZ2
  Y2=QFY(J,IE1)
  Y1=QFY(J,IE1)
  Y22=Y22(J,IE1)
  Y21=YT2(CB,IE1)
  HY=Y2-Y1
  A=CY22-Y12/(C.HD)
  B=0.5*HY12
  D=HY/H-HI=CY22+2.*HY12/6.
  DY=1
  YT2D=(CC(MA)+DELZ+B)+DELZ+C=DELZ+D
CONTINUE
CALL CUBIC1(X02,XX,YT,YT2)
DO 208 J=1,NZ2-1
  IF(ZZ1.GE.X0(J).AND.ZZ1.LE.X0(J+1)) 80 TO 218
CONTINUE
TYPE P,'OUT OF RANGE X0= ',XI
IF(X1.GT.X0(NZ2)) J=NZ2-1
IF(X1.GT.X0(NZ2)) DELX=X0(NZ2)-X0(NZ2-1)
IF(X1.LT.X0(1)) J=1
IF(X1.LT.X0(1)) DELX=0.0
Y1(CD)=1.E-18
80 TO 58
80 TO 211
DELX=X0-XX(J)
IB=J
IE=J+1
H=X0(CD)-X(CB)
HY=Y(T2(CD))-Y(T2(CB))
A=(YT2(CD)-YT2(CB))/(C.HD)
B=0.5*YT2(CB)
D=HY/H+HY(YT2(CD)+2.*YT2(CB))/6.
DY=YT2(CB)
Y1(CD)=CC(MA)+DELX+2.*B+D=DELX+C
CONTINUE
RETURN
END

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SUBROUTINE XMCL(XDX,Y,XH)
DIMENSION HC(20),HY(20),Y(20),Y2(20),XH(20)
COMMON /A/ NX,N1,N2,NZ,N21,N22,XL,T,X(21),Z(11),XC(20),ZC(18)
1,XX6,WS,XS,XE
C      TYPE P, 'X'
C      TYPE B, X
NN=N1\1
DO 10 I=1,NN
  II=I\1
  HC(I)=X(CI)\>-XC(I)
  HY(I)=Y(CI)\>-YC(I)
10 CONTINUE
CALL CUBIC(N1,X,Y,Y2)
S1=(YC(1)\>/HC(1)\>-HC(1))+(Y2(2)\>+2.*Y2(1))/6.
DH=X(1)\>-XS
DH1=X(1)\>-X(1)-XS+XS
IF(XDX.NE.0) XH(1)=S1+DH/2.+CY(1)-S1*X(1))+DH
DO 20 I=2,NI
  II=I\1
  A=(YC(1)\>+1)-Y2(1)\>/C8.+HC(1)\>
  B=0.5*Y2(1)\>
  C=(YC(1)\>/HC(1)\>-HC(1))+(Y2(1)\>+1)+2.*Y2(1)\>/6.
  D=Y(1)\>
  H1=HC(1)
  IF(XDX.EQ.0) XH(1)=C
  IF(XDX.NE.0) XH(1)=C+OM(1)\>+CCCCA/4.+H1+B/3.+H1+C/2.+H1
  1+DH\>H1
20 CONTINUE
DH=XS-X(H1)
S2=CC3\>+AD\>+B\>+B\>+D\>+C
IF(XDX.EQ.0) XH(N1)=S2
DH1=XS-X(H1)+X(N1)
IF(XDX.NE.0) XH(N1)=D+X(H1)+S2+DH/2.+CY(N1)-S2*X(N1))+DH
RETURN
END
SUBROUTINE INDU2
COMMON /A/ NX,N1,N2,NZ,N21,N22,XL,T,X(21),Z(11),XC(20),ZC(18)
1,XX6,WS,XS,XE
COMMON /C/ UTC20,10,10),UL(20,10,10)
COMMON /D/ V1C20,10),V2C20,10),V3C20,10)
Y=0.8
XX=X(CN1)
DO 20 L=1,NZ1
  ZZ=ZDLJ
  CALL S11(00,Y,ZZ)
  CALL S12(00,Y,ZZ)
  CALL S13(00,Y,ZZ)
  DO 30 JJ=1,NZ1
    DO 30 II=1,NX1
      ULCII,JJ,L)=V1CII,JJ)+V2CII,JJ
      UTCII,JJ,L)=ULCII,JJ,L)+V3CII,JJ
30 CONTINUE
20 CONTINUE
RETURN
END
SUBROUTINE MICHSS,CW
C*****
C      FUNCTION: THIS SUBROUTINE CALCULATES THE WAVE RESISTANCE COEFF.
C      BY MICHELL INTERVAL.
C*****
DIMENSION SS(20,10)
COMMON /A/ NX,N1,N2,NZ,N21,N22,XL,T,X(21),Z(11),XC(20),ZC(18)

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1, XK8, WS, XS, XE
COMMON /SM/ SH(28,18)
EXTERNAL RT
DO 1 I=1,NX1
    DO 1 J=1,NZ1
        SH(I,J)=SSCI(J)
CONTINUE
ST=0.0
EN=69.995*3.1416/180.
CALL SIMPSN(ST,EN,RT,VALD
CC=2./CWS*3.1416*WS*WS
CW=VALD*CC
RETURN
END

FUNCTION RT(THETA)
DIMENSION XC(28),YC(28),ZH(18),TEMP(28)
COMMON /A/ NX,NX1,NX2,NZ,NZ1,NZ2,XL,T,XC21,ZC11,XC28,ZC18
1, XK8, WS, XS, XE
COMMON /SM/ SH(28,18)
F1=COS(THETA)
B=XXB/F1
A=B/F1
ZH=Z(NZ)=A
D=EXP(C2N)
DO 18 I=1,NZ1
    K=NZ1+I-1
    T=AN(ZK)
    D1=EXP(T)
    ZH(K)=D-D1
    D=D1
18 CONTINUE
XH=X(NZ0)+B
D=SIN(XH)
E=COS(XH)
DO 28 I=1,NX1
    K=NX1+I-1
    T=BW(XK)
    D1=BN(T)
    E1=COST
    XH(K)=D-D1
    YH(K)=E-E1
    D=D1
    E=E1
28 CONTINUE
DO 38 I=1,NX1
    TEMP(I)=0.0
    DO 48 J=1,NZ1
        TEMP(I)=TEMP(I)+SH(I,J)*ZH(J)
48 CONTINUE
38 CONTINUE
P8=0.0
Q8=0.0
DO 58 I=1,NX1
    P8=P8+TEMP(I)*XH(I)
    Q8=Q8+TEMP(I)*YH(I)
58 CONTINUE
RT=(P8*P8+Q8*Q8)**0.5*PI**0.5
RETURN
END

SUBROUTINE WAVE(WS,SM,ETA,ETA1)
DIMENSION UC(28,18),ETAC(28),SH(28,18),ETA1(28),SM(28,18)
COMMON /A/ NX,NX1,NX2,NZ,NZ1,NZ2,XL,T,XC21,ZC11,XC28,ZC18

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1,XX0,WS,X5,XE
COMMON /D/ V1C(28,18),V2C(28,18),V3C(28,18)
XX=XC(NX1)
YY=0.0
ZZ=0.0
CALL SU1CX,YY,ZZ
CALL SU2CX,YY,ZZ
CALL SU3CX,YY,ZZ
DO 18 J=1,NX1
  V1C1,JJ=V1C1,JJ+V2C1,JJ
  V3C1,JJ=V1C1,JJ+V3C1,JJ
19  CONTINUE
DO 28 J=1,NX1
  ETAC(J)=0.0
  DO 38 II=1,NX1
    IS1=NX1-J-II
    IS2=NX1+J-II
    DO 38 JJ=1,NZ1
      IF(C1,LE,JJ) UC1,JJ=V3C1,JJ
      IF(C1,BT,JJ) UC1,JJ=-V1C1S2,JJ
38  CONTINUE
DO 48 II=1,NX1
  DO 48 JJ=1,NZ1
    ETAC(J)=ETAC(J)+UC1,JJ+SMC1,JJ
48  CONTINUE
29  CONTINUE
DO 58 I=1,NX1
  ETAC(I)=2.*ETA1C1
58  CONTINUE
DO 120 J=1,NX1
  ETAC(J)=0.0
  DO 138 II=1,NX1
    IS1=NX1-J-II
    IS2=NX1+J-II
    DO 138 JJ=1,NZ1
      IF(C1,LE,JJ) UC1,JJ=V3C1S1,JJ
      IF(C1,BT,JJ) UC1,JJ=-V1C1S2,JJ
138 CONTINUE
DO 148 II=1,NX1
  DO 148 JJ=1,NZ1
    ETA1C1=ETA1C1+UC1,JJ+SMC1C1,JJ
148 CONTINUE
128 CONTINUE
DO 168 I=1,NX1
  ETA1C1=2.*ETA1C1
168 CONTINUE
RETURN
END
SUBROUTINE SU1CX,Y,ZZ
COMMON /A/ NX,NX1,NX2,NZ,NZ1,NZ2,XL,T,XC(1),ZC(1),XC(28),ZC(18)
1,XX0,WS,X5,XE
COMMON /D/ V1C(28,18),V2C(28,18),V3C(28,18)
COMMON /F/ FC(1,11)
DOUBLE PRECISION F1,F
DO 18 J=1,NZ
  Z1=ZC(J)
  DO 18 I=1,NX
    XI=XC(I)
    CALL COEF1CX,Y,ZZ,XI,Z1,XX0,F1
    FCI,JJ=F1
18  CONTINUE

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CC=1./C4.*3.1416
DO 98 J=1,NZ1
JI=J+1
DO 98 I=1,NX1
II=I+1
P=FC(I,J)+FC(I,J)-FC(I,J)
V1(I,J)=P*CC
38 CONTINUE
RETURN
END
SUBROUTINE COEF0(X,Y,ZZ,X1,Z1,XX0,F1)
DOUBLE PRECISION R1,R2,F1,D1,D2,XY,YZ
DOUBLE PRECISION DX1,C1,C2
DX1=X1-XX
C1=Z1-ZZ
C2=Z1-ZZ
IF(C1.EQ.0.BD C1=-1.E-4
IF(C2.EQ.0.BD C2=-1.E-4
XY=DX1+Y*Y
R1=SQRT(XY+C1*C1)
R2=SQRT(XY+C2*C2)
IF(XY.LE.1.E-18) F1=DL08(ABSC(C2))/ABSC(C1)
D1=C1+R1
D2=C2+R2
IF(XY.GT.1.E-18) F1=DL08(D1/D2)
RETURN
END
SUBROUTINE SU20(X,Y,ZZ)
COMMON /A/ NX,NX1,NX2,NZ,NZ1,NZ2,XL,T,X(21),Z(11),XC(28),ZC(18)
1,XX0,W5,X5,XE
COMMON /D/ V1(28,18),V2(28,18),VS(28,18)
COMMON /F/ FC21,11
DO 18 J=1,NZ
Z1=ZC(J)
DO 18 I=1,NX
TYPE='SU2' J=I,J,I=I,I
XI=X(I)
CALL COEF0(X,Y,ZZ,X1,Z1,XX0,F1)
FC1,J=F1
18 CONTINUE
CC=-1./C4.*3.1416*3.1416
DO 98 J=1,NZ1
JI=J+1
DO 98 I=1,NX1
II=I+1
P=FC(I,J)+FC(I,J)-FC(I,J)
V2(I,J)=P*CC
38 CONTINUE
RETURN
END
SUBROUTINE COEF0(X,Y,ZZ,X1,Z1,XX0,F1)
DIMENSION SF1(3),SF2(3)
COMMON /A/ Y1,XK,DX1,C1
Y1=Y
XX=XX0
DX1=XX-X1
C1=Z1-ZZ
Y1=Y
XX=XX0
ST=0.8
EN=89.005*3.1416/188.
NI=88

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D1=(CEN-ST0)/N1
DO 18 J=1,3
  SF1(J)=0.0
CONTINUE
DO 28 I=1,N1+
  THETA=ST0*D1*(I-1)
  CALL CONCTHETA,F
  II=I/2#2
  IF(I.EQ.1.OR.I.EQ.N1+1) ID=1
  IF(II.EQ.1) ID=2
  IF(II.NE.1.AND.I.NE.1.AND.I.NE.N1+1) ID=3
  SF1(ID)=SF1(ID)+F
CONTINUE
FSUM1=CSF1(ID)*4.+SF1(2)*2.+SF1(3)*D1/3.
FSUM2=FSUM1
IDC=1
IF(IDC.EQ.1) GO TO 110
K=1
K=K+1
SF2(1)=SF1(1)
SF2(3)=SF1(2)+SF1(3)
N2=N1
D2=D1/2.
SF2(2)=0.0
DO 78 I=1,N2
  THETA=ST0*D2*(I-1)
  CALL CONCTHETA,F
  SF2(I)=SF2(I)+F
CONTINUE
FSUM2=CSF2(1)*4.+SF2(2)*2.+SF2(3)*D2/3.
IF(CSUM1.EQ.0.0) CXF=0.0
IF(CSUM1.NE.0.0) CXF=ABSCFSUM2/FSUM1-1.0
IF(CXF.LE.1.E-3) GO TO 110
FSUM1=FSUM2
DO 100 J=1,3
  SF1(J)=SF2(J).
CONTINUE
N1=2.*N2
D1=D2
IF(K.GT.73) GO TO 110
GO TO 46
C
110 F1=FSUM2
RETURN
END
SUBROUTINE CONCTHETA,YF
DIMENSION BC23
COMMON /A1/Y,XKB,DX1,C1
T=TAN(CTHETA)
FF1=COS(CTHETA)
CO1=FF1*FF1
F2=SIN(CTHETA)
F3=Y*F2
A=XKB/CO1
BC13=DX1*FF1+F3
BC23=DX1*FF1-F3
XT=C1*A
YF=0.0
IF(CABS(CYD).LE.1.E-10) IS=1
IF(CABS(CYD).GT.1.E-10) IS=2
DO 28 I=1,IS
  YT=BC13*XA

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FS=XT+YT+YT+YT
GS=XT+0.82*XT+XT+0.82*YT+YT
IF(FS.GT.1.0 AND GS.GT.0.0) CALL SIMPCXT,YT,VAL,VALID
IF(FS.LE.1.0 AND GS.LE.0.0) CALL SCOMXT,YT,VAL,VALID
YF1=2.*VAL+ALDBCC1=C1+B1*D(B1)(I)
YF=YF+YF1
28 CONTINUE
IF(C1.EQ.1.0 YF=2.*YF
RETURN
END
SUBROUTINE SIMPCCT1,TB1,VALR,VALID
DIMENSION S1(3),S2(3),SA1(3),SA2(3),CX(2)
NI=28
D1=3.1416/C2.NI
K=1
DO 10 I=1,3
  S1(I)=0.0
  SA1(I)=0.0
10 CONTINUE
DO 20 I=1,NI+1
  ALPHA=D1*(I-1)
  IF(I.EQ.NI+1) ALPHA=60.006*3.1416/180.
  T=TAN(ALPHA)
  CS=COS(ALPHA)
  AI=T*TC1
  XII=EXP(-T)/CCS=CS
  XIR=A1*XII
  XII=TB1*XII
  XII2=CA1*AI+CTB1*TB1
  VS=XIR/XII
  VS1=XII/XII2
  II=I/2
  IF(I.EQ.1.OR.I.EQ.NI+1) ID=1
  IF(I.EQ.2) ID=2
  IF(I.EQ.1.OR.I.EQ.NI+1.AND.I.NE.1.AND.I.NE.NI+1) ID=3
  S1(I)=S1(I)+VS
  SA1(I)=SA1(I)+VS1
20 CONTINUE
VALT=(SA1(I)+4.*SA1(2)+2.*SA1(3))/D1/3.
VAL=(SA1(I)+4.*SA1(2)+2.*SA1(3))/D1/3.
45 K=K+1
  S2(I)=S1(I)
  S2(2)=S1(2)+S1(3)
  SA2(I)=SA1(I)
  SA2(2)=SA1(2)+SA1(3)
  N2=NI
  D2=D1/2.
  S2(2)=0.0-
  SA2(2)=0.0
  DO 70 J=1,N2
    ALPHA=D2*(2*I-1)
    T=TAN(ALPHA)
    CS=COS(ALPHA)
    AI=T*TC1
    XII=EXP(-T)/CCS=CS
    XIR=A1*XII
    XII=TB1*XII
    XII2=CA1*AI+CTB1*TB1
    VS=XIR/XII
    VS1=XII/XII2
    S2(J)=S2(2)-VS
    SA2(J)=SA2(2)+VS1
70 CONTINUE

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78    CONTINUE
      VAL2=(S2(1)+4.*S2(2)+2.*S2(3))/D2/3.
      VA2=(SA2(1)+4.*SA2(2)+2.*SA2(3))/D2/3.
      IF(CVAL1.EQ.0.0) CX(1)=0.0
      IF(CVAL1.NE.0.0) CX(1)=ABSCVAL2/VAL1-1.0
      IF(CVA1.EQ.0.0) CX(2)=0.0
      IF(CVA1.NE.0.0) CX(2)=ABSCVA2/VA1-1.0
      IF(CCX(1).LE.1.E-3.AND.CX(2).LE.1.E-3) GO TO 110
      VAL1=VAL2
      VA1=VA2
      DO 100 J=1,3
        S1(J)=S2(J)
        S1(J)=SA2(J)
100   CONTINUE
      N1=2*N2
      D1=D2
      IF(CK.GT.100.00) GO TO 110
      GO TO 45
C
110   VALR=VAL2
      VALI=VA2
      RETURN
      END
      SUBROUTINE SCOMCXT,YT,VALR,VALI
      COMPLEX# Z,VN,ZI
      VNR=0.0
      VN=0.0
      Z=CHPLXCT,YT)
      R=ABSCZ
      IF(CXT.NE.0.0) TH=ATAN(CABSCT/XT))
      IF(CXT.EQ.0.0) TH=0.1416/2.
      IF(CXT.GE.0.0.AND.YT.GT.0.0) TH=TH+0.0
      IF(CXT.GE.0.0.AND.YT.LT.0.0) TH=-TH
      IF(CXT.LT.0.0.AND.YT.GT.0.0) TH=0.1416-TH
      IF(CXT.LT.0.0.AND.YT.LT.0.0) TH=-0.1416+TH
      IF(YT.EQ.0.0) GO TO 110
      ZI=Z
      VN=0.5772157-CHPLX(CAL08(R),TH)-ZI
      DO 20 I=2,1000
        XN=FLOAT(I)
        ZI=CXH(1)*ZI-C-Z3/C0H(0)
        VN=VN-ZI
        ARL=REAL(VN)
        AIM=AIMREAL(VN)
        IF(CARL.EQ.0.0.AND.AIM.EQ.0.0) GO TO 25
        BRL=ABSC1.-VNR/ARL)
        BIM=ABSC1.-VN/ARL)
        IF(CABS(BRL).LE.1.E-5.AND.CABS(BIM).LE.1.E-5) VN=VN*EXP(Z)
        IF(CABS(BRL).LE.1.E-4.AND.CABS(BIM).LE.1.E-5) GO TO 100
        VNR=ARL
        VN=AIM
20    CONTINUE
      VN=VN*EXP(Z)
      WRITE(6,*) 'UNCONVERGENCE Z=',Z
      GO TO 100
110  X1=ABSCXT
      VN=0.5772157-ALOG(X1)-X1
      DO 30 I=2,1000
        XN=FLOAT(I)
        X1=CXH(1)*X1+C0H(0)
        VN=VN-X1
        ARL=REAL(VN)

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```

AIM=ADMASCVND
BRL=ABS(C1-.ARL/VNRD)
BIM=ABS(C1-.AIM/VND)
IF(CABS(BRL).LE.1.E-3.AND.ABS(CBIM).LE.1.E-3) VN=VN*EXP(Z)
IF(CABS(BRL).LE.1.E-3.AND.ABS(BIM).LE.1.E-3) GO TO 100
VNR=ARL
VND=AZH
38 CONTINUE
TYPE *,'UNCONVERGENCE X=',X
VN=VN*EXP(Z)
100 VALRMREALCVND
VALI=ADMASCVND
RETURN
END
SUBROUTINE SUS(X0,X,Y,ZD)
COMMON /A/ NX,NX1,NX2,NZ,NZ1,NZ2,XL,T,XC210,ZC110,XCC200,ZCC100
1,XGB,NS,XS,XE
COMMON /D/ V1C20,180,V2C20,180,V3C20,180
COMMON /F/ F(21,11)
COMMON /A1/ Y1,X0,DX1,C1,II
Y1=Y
X0=XGB
DO 18 J=1,NZ
  (Z1=ZCJ)
  DO 18 I=1,NX
    TYPE *,SUS 3=J,I,  I=I,I
    X1=XCID
    DX1=XX-X1
    C1=ZZ+Z1
    IF(CABS(CC1).LE.1.E-6) C1=-1.E-4
    IF(CABSY2.GT.1.E-180) CALL INT1(GF1)
    IF(CABSY2.LE.1.E-180) CALL INT2(GF1)
    FC1,J=I
18 CONTINUE
CC=1./3.1416
DO 38 J=1,NZ
  J1=J+1
  DO 38 I=1,NX1
    II=I+1
    P=FC1(J1)+FC1(JD)-FC1(J1)-FC1(JD)
    V3C1(JD)=P*CC
38 CONTINUE
CONTINUE
RETURN
END
SUBROUTINE INT2(GF1)
COMMON /A1/ Y,X0,X1,C1,II
EXTERNAL FU
IF(DX1.LE.8.E-8) DX1=8.E-8
SUM=0.8
SUM1=0.8
TH1=0.8
DO 18 I=1,1000
  SC=FLOAT(C1)
  S=X0*X1/CSC=3.1416
  IF(S.GT.1) GO TO 18
  TH2=ACOS(S)
  IF(TH2.GT.3.1416/2.) TH2=3.1416/2.
  CALL SIMPSN(TH1,TH2,FU,VAL)
  SUM=SUM+VAL
  IF(TH2.GE.3.1416/2.) GO TO 28
  IF(CSUM.EQ.0.8) CX=0.8
  28

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```

IF(SUM.NE.0.0) CX=ABSCSUM1/SUM-1.0
IF(CX.LE.1.E-3) GO TO 28
SUM1+SUM
TH1=TH2
18 CONTINUE
ANG=TH2/3.1416*180.
F1+SUM*2.
RETURN
END

SUBROUTINE INT1(F1)
DIMENSION TC100
COMMON /A1/X,XKB,DX1,C1,I1
EXTERNAL F1
FC0=X*180./3.1416
GCA,B,THETA=A-COS(THETA)-B-TAN(THETA)
PI=3.1416
B=DX1/Y
R=SQRT(DX1*X+Y*Y)
ALPHA=A-COS(DX1/Y)
AA=FCTH1
THF=PI/2.
F1=0.0
TM=-PI/2.+ALPHA
IF(TH1.GT.0.0) GO TO 28
TT=FCTH1
SUM1=0.0
N=0
THETA=-89.*3.1416/180.
19 N=N+1
BB=FLOAT(N)
A=BB*PI/CXKB*Y
5 TH2=ATAN(A*COS(THETA)-B)
IF(CABS(THETA/TH2-1.).LE.1.E-2) TS=(THETA+TH2)/2.
IF(CABS(THETA/TH2-1.).LE.1.E-2) CALL STACKN(TS,T)
IF(CABS(THETA/TH2-1.).LE.1.E-2) GO TO 18
THETA=(TH2+THETA)/2.
IF(THETA.LT.0.0) GO TO 5
N=N-1
IF(CN.EQ.0) N=1
IF(CN.GT.100) NN=100
IF(CN.LE.100) NN=N
AU=0.0
DO 15 I=1,NN
N1=NN-I-1
AL=TCN1
IF(CAL.LT.TH1) AL=TH1
IF(CN.EQ.1) AL=TH1
CALL SIMPSICAL,AU,F1,VALD
A1=FCAL2
A2=FCAL3
F1=F1+VALD
IF(CABS(CAL-TH1).LE.1.E-6) GO TO 28
IF(CN.EQ.0.0) CX=0.0
IF(C1.NE.0.0) CX=ABSCSUM1/F1-1.
IF(CX.LE.1.E-4) GO TO 20
SUM1=F1
AU=AL
15 CONTINUE
20 IF(TH1.LT.0.0) TH1=0.0
XL=TH1
XU=-89.00*3.1416/180.
BB=FLOAT(N)

```

```

A=BB*PI/(CX*Y)
25 XH=XL+XU/2.
YL=GGCA,B,XL)
YU=GGCA,B,XU)
YH=GGCA,B,XH)
IF CYMH>YL .0E-.0) XL=XH
IF CYMH<YL .LT.-.0) XU=XH
TH=FOOL
T2=FOUD
IF CABSKU-XL).LE.1.E-6) TH2=(XL+XU)/2.
IF CABSKU-XL).LE.1.E-6) GO TO 38
GO TO 25
38 IF CTH2.LT.TH1) GO TO 58
CALL SIMPSN(TH1,TH2,FUI,VAL)
A1=CTH1
A2=CTH2
F1=F1+VAL
IF CTH1.GT.FC88) .AND. ABS(VAL).LE.1.E-20) GO TO 79
IF C1.EQ.0.0) GO TO 55
IF C1.NE.0.0) CX=ABSCSUM1/F1-1.
IF CX.LE.1.E-4) GO TO 78
SUM1=F1
55 TH1=TH2
56 N=N+1
XL=TH1
IF CN.LE.1000) GO TO 23
78 RETURN
END.
SUBROUTINE STACK(N,T,T0)
DIMENSION T(10)
IF CN.LE.10) GO TO 18
DO 28 I=1,9
   II=I+1
   TCD=T(I)
28 CONTINUE
TC10=TS
GO TO 38
38 TN0=TS
RETURN
END
SUBROUTINE SIMPSN(A,B,F,VAL)
DIMENSION S1(3),S2(3)
NI=18
D1=(B-A)/NI
K=1
DO 19 J=1,3
   S1(J)=0.8
19 CONTINUE
DO 20 I=1,NI+1
   ALPHAM=A+D1*(I-1)
   V8=FCALPHA
   II=I/2+2
   IF CI.EQ.1 .OR. I.EQ.NI+1) ID=1
   IF CI.EQ.2) ID=2
   IF CI.NE.1 .AND. I.NE.1 .AND. I.NE.NI+1) ID=3
   S1(ID)=S1(ID)+V8
20 CONTINUE
VAL=(S1(1)+4.*S1(2)+2.*S1(3))*Q1/3.
46 K=K+1
S2(1)=S1(1)
S2(3)=S1(2)+S1(3)
N2=NI

```

```

D2=01/2.
S2C2=0.8
DO 70 I=1,N2
  ALPHA=A+D2*(2*I-1)
  VS=FC(ALPHA)
  S2C2=S2C2+VS
70 CONTINUE
VAL2=(S2C1)+4.*S2(2)*2.+S2(3))*D2/3.
IF(CVAL1.EQ.0.0) CX=0.0
IF(CVAL1.NE.0.0) CX=ABS(CVAL2/VAL1-1.)
IF(CX.LE.1.E-4) GO TO 118
VAL1=VAL2
DO 100 J=1,3
  S1CJ=S2CJ
100 CONTINUE
N1=2*N2
D1=D2
IF(CX.GT.1.0) GO TO 118
GO TO 45
C
118 VAL=VAL2
200 RETURN
END
FUNCTION FUC(THETA)
DOUBLE PRECISION XI,F1,F2,B1,B2,A
COMMON /A/ Y,XKB,DX1,C1,I
F1=COS(THETA)
F2=SIN(THETA)
A=XKB/(F1+F2)
XI=EXP(A*C1)
B1=XKB*DX1/F1
FU=XI*SINC(B1)
RETURN
END
FUNCTION FU1(THETA)
COMMON /A/ Y,XKB,DX1,C1,I
F1=COS(THETA)
F2=SIN(THETA)
A=XKB/(F1+F2)
XI=EXP(A*C1)
B1=A*DX1+F1+Y*F2
FU=XI*SINK(B1)
RETURN
END
SUBROUTINE CUBIC1(N,X,Y,Y2)
DIMENSION XCN1,YCN1,Y2CN1,FC301,GC301
Y2C1=0.8
Y2CN1=0.8
N1=N-1
S1C1=0.
FC1=0.
DO 2 K=1,N1
  J2=K+1
  H2=XC(J2)-XG(J2)
  R2=(Y(J2)-Y(K))/H2
  IF(K.EQ.1) GO TO 1
  Z=1./((H1+H2)-H1*GC1)
  GC1=Z*H2
  H=0.1*RC2-R1
  IF(CX.EQ.0.2) H=H-1.1*Y2C1
  IF(CX.EQ.1.0) H=1.1*H2*Y2CN1
  FC0=Z*(H-H1)*FC1
  FC1=FC0
  GC1=GC1+1.
  GO TO 2
1   H=0.1*RC2-R1
  IF(CX.EQ.0.2) H=H-1.1*Y2C1
  IF(CX.EQ.1.0) H=1.1*H2*Y2CN1
  FC0=Z*(H-H1)*FC1
  FC1=FC0
  GC1=GC1+1.
  GO TO 1
  
```

```

1      J1=K
      H1=H2
      R1=R2
2      CONTINUE
      Y2(N1)=F(Y1(N1))
      IF(N1.LE.2) RETURN
      N2=N1-1
      DO 3 S(J1)=N2
           K=N1-J1
           Y2(K)=F(Y1(K)-B(K))*Y2(K+1)
3      CONTINUE
      RETURN
      END

```

```
C=====
C PROGRAM VLCTY
C
```

THIS PROGRAM CALCULATES THE NONDIMENSIONAL VELOCITY COMPONENTS
ALONG THE ISOBARS WHICH ARE IN THE OUTPUT DATA OF PROGRAM "THNGU".

INPUT DATA:

- (1) FRDULE NUMBER, DRAFT AND LBP OF THE SHIP
- (2) SOURCE STRENGTH OF EACH SOURCE PANEL WHICH IS ALSO IN THE OUTPUT
DATA OF PROGRAM "THNGUC".

```
C=====
C
C DIMENSION FILE(4),UUC200,UK20,18),MW(20),SH(20,18),FIL1(4)
COMMON //A/NX,NX1,NX2,NZ,NZ1,NZ2,XL,T,XC210,Z(115),XC(28),ZC(18)
I,FR,VEL,XS,XE,XNB
COMMON /D/V1C28,10,3),V2C28,10,3),V3C28,10,3)
TYPE *, 'ENTER OUTPUTFILE NAME'
ACCEPT 1,FILE
FORMAT(444)
CALL ASSIGNC1,FILE)
TYPE *, 'ENTER INPUTFILE NAME'
ACCEPT 1,FIL1
CALL ASSIGNC2,FIL1)
XL=2.0
NX=21
NZ=11
NX1=NX-1
NZ1=NZ-1.
READ(2,*),FR,T,D,XLL
DO 3 I=1,NX1
  READ(2,*),CSMC1,I,J=1,NZ1)
3 CONTINUE
CALL CLOSE(2)
T=2.0/TD*XLL
X0G=1.0/(2.0*FR*T)
WRITE(1,*),'FN = ',FR
CALL POINT
DO 188 K=1,NZ1
  ZZ=ZCK(K)
  XX=XC(NX1)
  ICP=3
  YY=0.0
  CALL SUP1(CXX,YY,ZZ,ICP)
  CALL SUP2(CXX,YY,ZZ,ICP)
  CALL SUP3(CXX,YY,ZZ,ICP)
  DO 18 I=1,NX1
    DO 18 J=1,NZ1
      V1C1,I,J=V1C1,I,J+V2C1,I,J,13
      V2C1,I,J=V1C1,I,J+V3C1,I,J,13
      V1C1,I,33=V1C1,I,J,33+V2C1,I,J,33
      V3C1,I,33=V1C1,I,J,33+V3C1,I,J,33
18 CONTINUE
  DO 28 J=1,NX1
    UK(J)=0.0
  DO 38 II=1,NX1
    ISI=NX1-J+II
    IS2=NX1-J-II
    DO 38 JJ=1,NZ1
      IFCII,LE,J,D UCII,J,JD=V3CIS1,II,J,13
      IFCII,BT,J,D UCII,J,JD=V1CIS2,II,J,13
38
```

```

38      CONTINUE
      DO 49 II=1,NX1
      DO 49 JJ=1,NZ1
         UU(JJ)=UU(JJ)+UC(II,JJ)*SM(II,JJ)
48      CONTINUE
49      CONTINUE
      DO 58 J=1,NX1
         VV(JJ)=0.0
      DO 68 II=1,NX1
         ISI=NX1-II
         IS2=NX1+II
         DO 68 JJ=1,NZ1
            IF(II.LE.JJ) UC(II,JJ)=V3(ISI,JJ,3)
            IF(II.GT.JJ) UC(II,JJ)=V1(IS2,JJ,3)
68      CONTINUE
      DO 78 II=1,NX1
      DO 78 JJ=1,NZ1
         VV(JJ)=VV(JJ)+UC(II,JJ)*SM(II,JJ)
78      CONTINUE
58      CONTINUE
      WRITE(1,*) /
      WRITE(1,*) /
      WRITE(1,*) 'ISOBAR No. ',K
      WRITE(1,*) /
      WRITE(1,*) 'NONDIMENSIONAL DISTURBANCE VELOCITY u/U'
      WRITE(1,*) CU(1,I,NX1)
      WRITE(1,*) /
      WRITE(1,*) 'NONDIMENSIONAL DISTURBANCE VELOCITY w/U'
      WRITE(1,*) CW(1,I,NX1)
      WRITE(1,*) /
100     CONTINUE
      STOP
      END
      SUBROUTINE POINT
      COMMON /A/ NX,NX1,NX2,NZ,NZ1,NZ2,XL,T,X(21),Z(11),XCC20,ZC(10)
      1,FR,VEL,XS,XE,XKB
      SX-XL/CNX-1
      SZ=T/(CNZ-1)
      DO 18 I=1,NX
         XC(I)=1.+SZ*(CI-1)
         IF(CI.EQ.NZ) GO TO 18
         XC(I)=1.+SZ*(CI-0.5)
18      CONTINUE
      DO 28 I=1,NZ
         Z(I)=T+SZ*(CI-1)
         IF(CI.EQ.NZ) GO TO 28
         Z(I)=T+SZ*(CI-0.5)
28      CONTINUE
      RETURN
      END
      SUBROUTINE SUP1(XY,ZZ,ICP)
      COMMON /A/ NX,NX1,NX2,NZ,NZ1,NZ2,XL,T,X(21),Z(11),XCC20,ZC(10)
      1,FR,VEL,XS,XE,XKB
      COMMON /D/ V1(20,10,3),V2(20,10,3),V3(20,10,3)
      COMMON /F/ F(21,11,3),FI(33)
      DOUBLE PRECISION FI,F
      DO 18 J=1,NZ
         ZI=Z(J)
      DO 18 I=1,NX
         XI=X(CI)
         CALL COEF1(XY,ZZ,XI,ZI,XKB,FI,ICP)
         TYPE *,FI
C

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```

      DO 28 II=1,ICP
         F(I,J,II)=F1(II)
      CONTINUE
  28 CONTINUE
  18 CC=1./C4.*S.1416)
      DO 38 J=1,NZ1
         J1=J+1
      DO 38 I=1,NX1
         I1=I+1
      DO 48 II=1,ICP
         P=F(I1,J1,II)+F(I,J,II)-F(I,J1,II)-F(I1,J,II)
         V(I,J,II)=P*CC
  48 CONTINUE
  38 CONTINUE
      RETURN
     END

SUBROUTINE COEF(CXX,Y,ZZ,XI,ZI,XX0,FI,ICP)
DIMENSION FI(3)
DOUBLE PRECISION R1,R2,FI,D1,D2,XY,YZ
DOUBLE PRECISION DX1,C1,C2
DX1=XI-XX
C1=ZI-ZZ
C2=ZI+ZZ
IF(C1.EQ.0.0) C1=1.E-4
IF(C2.EQ.0.0) C2=1.E-4
XY=DX1+Y*WY
R1=SQRT(XY+C1*C1)
R2=SQRT(XY+C2*C2)
IF(ABS(CY).LE.1.E-10) FI(2)=0.0
IF(ABS(CY).GT.1.E-10) FI(2)=ATAN(CDX1+C1)/CY*R1
D1=ATAN(CDX1+C2)/CY*R2
IF(CY.LE.1.E-10) FI(1)=DL0G(CABS(C2)/ABS(C1))
D1=C1+R1
D2=C2+R2
IF(CY.GT.1.E-10) FI(1)=DL0G(C1/D2)
YZ=Y*WY+C1*C1
IF(CY.LE.1.E-10) FI(3)=2.*DL0G(CABS(CX1))
D1=DX1+R1
D2=DX1+R2
IF(CY.GT.1.E-10) FI(3)=DL0G(CD1*D2)
IF(CICP.EQ.1) FI(2)=0.0
IF(CICP.EQ.1) FI(3)=0.0
      RETURN
     END

SUBROUTINE SUP2(CXX,Y,ZZ,ICP)
COMMON //A/NX,NX1,NX2,NZ,NZ1,NZ2,XL,T,X(2),Z(2),XCC(28),ZCC(18)
1,FR,VEL,XS,XE,XK0
COMMON //D/V1(28,18,3),V2(28,10,3),V3(28,10,3)
COMMON //F/F2(1,11,3),F1(3)
DO 19 J=1,NZ
   Z=Z(J)
DO 18 I=1,NX
   TYPE N,'SU2 J=1,J1' I=1,I
   XI=X(I)
   CALL COEF(CXX,Y,ZZ,XI,ZI,XX0,FI,ICP)
   DO 28 II=1,ICP
      F(I,J,II)=F1(II)
  28 CONTINUE
  18 CONTINUE
  19 CC=-1./C4.*S.1416+S.1416)
  20 DO 38 J=1,NZ1
     J1=J+1

```

```

DO 38 I=1,NK1
  I1=I+1,ICP
  DO 48 J=I,I+1
    G=M*(C1,J,I)+F(C1,J,IID)-F(I,J1,IID)*F(I,J)
    V=V+G,J,IID=M*C1
    IF(I>J,IID,EQ.33) V=V*G,J,IID-2.*V*G,J,IID
  CONTINUE
38 CONTINUE
RETURN
END
SUBROUTINE COEF(X,Y,Z,W1,Z1,X008,F1,ICP)
DIMENSION F1(10),YCFSD,THC4D
COMMON /A1/Y1,X008,C1
Y1=Y
XX=X008
DO 49 I=1,4
  C1=I*21
  FFC1=EQ.8.8, C1=-1.E-4
  THC1=-B.8
  THC2=-B.8,9.1416/188.
  THC3=-B.8,9563.1416/188.
  DO 19 I=1,ICP
    F1(I)=C1*THC1
    F1(I+1)=C1*THC2
    F1(I+2)=C1*THC3
 19 CONTINUE
49 CONTINUE
CALL IT8CTHETA2
THC4D=THETA
NM=4
CALL COMPCON1,TH
DO 58 I=1,3
  ST=THC2D
  EN=THC3D+1
  IF(ST,ED,END GO TO 28
  NM=8
  Z=0.
  CALL INTON,ST,EN,Y,F1,ICP,ICD
  DO 58 J=1,ICP
    F1(J)=Y+F1(J)
 58 CONTINUE
28 CONTINUE
58 CONTINUE
59 RETURN
END
SUBROUTINE COMPCON1,X0
DIMENSION X0(10)
C TYPE *.X
DO 5 J=1,N-1
  X0(J)=0.
  DO 19 I=1,X
    IF(X0(I).LE..X0(I+1)) GO TO 19
    TEMP=X0(I)
    X0(I)=X0(I+1)
    X0(I+1)=TEMP
 19 CONTINUE
CONTINUE
C CONTINUE
TYPE *.X
RETURN
END
SUBROUTINE IT8CTHETA2
COMMON /A1/Y,X008,X01,C1
THEND=-B.8,9563.1416/188.
THET=18.,W3,B.8,9.1416/188.

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```

100      ICONT=1
101      F1=COS(THET)
102      F2=SIN(THET)
103      A=XXB/(F1+F2)
104      BC1=OX1+YWF2
105      BC2=OX1-YWF2
106      IF(BC1>=BC2).LT.0.0 THETA=THEND
107      IF(BC1>=BC2).LT.0.0 GO TO 108
108      IF(BC1>=0.0E-03) TI=3.1416E2.
109      IF(BC1).LT.0.0D0 TI=3.1416E2.
110      XTH=C1*A
111      TEMP=0.
112      DO 48 I=1,2
113      YT=B(C1)*A
114      FS=XT*YT+YT*YT
115      GS=XT*0.82*XT*YT+0.82*YT*YT
116      IF(CFS.GT.1.AND.GS.GT.0.0 CALL S2MPCT,YT,VAL,VAL1)
117      IF(CFS.LE.1.0R.GS.LE.0.0 CALL S2MPCXT,YT,VAL,VAL1)
118      TEMP=TEMP+(ATAN(BC1/C1)+VAL1)/TI
119      CONTINUE
120      A1=ALOG(TEMP)/CI
121      CX1=ABSC(A1/A-1.0
122      A=A1
123      IF(CX1.GT.1.E-4) GO TO 30
124      A2=XXB/A1
125      IF(CABS(CX2).GT.1.0 THETA=THEND
126      IF(CABS(CX2).LE.1.0 THETA=ACOS(SQRT(CX2))
127      CX2=ABSC(THETA/THET-1.0
128      THET=THETA
129      ICONT=ICONT+
130      IF(ICONT.GT.50) GO TO 100
131      IF(CTHETA.NE.THEND.AND.CX2.GT.1.E-4) GO TO 20
132      RETURN
133      END
134      SUBROUTINE INTNCI(ST,EN,YF,ICP,ICC)
135      DIMENSION YFC3D,FCS3,SF1C3,S3,SP2C3,S3,FSUM1C3,FSUM2C3,CXF3D
136      COMMON /A1/Y,XXB,DX1,C1
137      WRITE(4,*) /
138      D1=(EN-ST)/N
139      DO 18 I=1,IC
140      DO 18 J=1,3
141      SF1C(I,J)=0.0
142      CONTINUE
143      DO 20 I=1,NI+1
144      THETA=ST+D1*(I-1)
145      ANG=THETA*180./3.1416
146      CALL CONKTHETA,F,ICP
147      WRITE(4,*) ANG,F
148      II=I/2#2
149      IF(I.EQ.1.OR.I.EQ.NI+1) IC=1
150      IF(I.EQ.1) IC=2
151      IF(I.EQ.NI) IC=3
152      IF(II.NE.I.AND.I.NE.1.AND.I.NE.NI+1) IC=3
153      DO 30 J=1,IC
154      SF1(J,IC)=SF1(J,IC)+FCJ
155      CONTINUE
156      CONTINUE
157      DO 48 I=1,IC
158      FSUM1C3=(SF1C(I,1)+4.*SF1C(I,2)+2.*SF1C(I,3))/D1/3.
159      FSUM2C3=FSUM1C3
160      CONTINUE
161      TYPE*,FSUM1
162      IF(ICC.EQ.1) GO TO 110

```

```

      KK+1
  45  DO 68 I=1,ICP
    SF2(I,1)=SF1(I,1)
    SF2(I,2)=SF1(I,2)+SF1(I,3)
  48  CONTINUE
  49  NORM
  50  DO=1/2.
  51  DO 68 I=1,ICP
    SF2(I,2)=0.8
  52  CONTINUE
  53  DO 78 I=1,N2
    THETA=ATAN2(SUM2(I-1),
    CONTHETA,Y,ICP)
    DO 68 J=1,ICP
    SF2(J,2)=SF2(J,1)*FC(I,J)
  54  CONTINUE
  55  CONTINUE
  56  DO 68 I=1,ICP
    SF3(I,1)=SF2(I,1)+4.*SF2(I,2)+SF2(I,3)*DO/3.
  57  IF(SUM1(I).EQ.8.8) FC(I)=8.8
  58  IF(SUM1(I).NE.8.8) FC(I)=ABS(SUM2(I))/SUM1(I)-1.
  59  IF(FC(I)>87.1.E-3) IRH=1
  60  CONTINUE
  61  TYPE*1000
  62  IF(KK.EQ.8) GO TO 118
  63  DO 100 J=1,3
    SUM1(I,J)=SF2(I,J)
  64  CONTINUE
  65  N1=N2-N2
  66  DIV=N2
  67  IF(KK.GT.7) GO TO 118
  68  GO TO 45
C
  118  DO 129 I=1,ICP
    YFC(I)=SUM2(I)
  129  CONTINUE
  130  RETURN
  131  END
  132  SUBROUTINE CONTHETA,Y,ICP,
  133  DIMENSION YFC(3),C02
  134  COMMON /A1/Y,X00,DX1,C1
  135  TANH(X00)
  136  PTCOS(TANH(X00))
  137  C02=PTCOS(TANH(X00))
  138  F2=H2INTHETAY
  139  FSWWF2
  140  AXXB/C02
  141  BC1=DX1*WTF1+F8
  142  BC2=DX1*WTF1+F9
  143  XH=C1*A
  144  DO 148 I=1,ICP
    YFC(I)=0.8
  145  CONTINUE
  146  IF(ABSY(Y).LE.1.E-03) I=1
  147  IF(ABSY(Y).GT.1.E-03) I=2
  148  DO 150 J=1,I
    Y+=C2*XJ
  149  E9=IXXT*Y+WFT
  150  E9=IXXT*Y+WFT

```

GS=XT+8.82*XT+XT+8.82=Y_T=YT
 IF(CS.GT.1.AND.GS.GT.8.) CALL SIMPCXT,YT,VAL,VALID
 IF(CS.LE.1.OR.GS.LE.8.) CALL SCOMCXT,YT,VAL,VALID
 YF1=2.*VAL+AL08(C1=C1+BK1)*BK1
 YFC1>YFC1+YF1
 IFC(IOP.EQ.1) GO TO 28
 IFCIS.EQ.2.AND.I.EQ.1) YFC2>YF1
 IFCIS.EQ.2.AND.I.EQ.2) YFC2>YFC2-YF1
 IFCIS.EQ.1) YF2>0.8
 IF(YT.LT.0.82 TI=3.1416
 IF(YT.GT.8.82 TI=8.1416
 YGS=ATAN(BK1/C1)+TI=EXP(CT)
 YFC3>YFC3+VAL(+YGS
 28 CONTINUE
 YFC2>YFC2+YF1
 YFC3>YFC3/FF1
 IFCIS.EQ.2) GO TO 388
 DO 38 I=1,ICP
 YFCID=2.*YFCID
 30 CONTINUE
 RETURN
 END
 SUBROUTINE SIMPCXT,TB1,VALR,VALID
 DIMENSION S1(3),S2(3),SA1(3),SA2(3),CX(2)
 NI=28
 D1=3.1416/(C2.*NI)
 K=1
 DO 10 I=1,3
 S1(CD)=0.8
 SA1(CD)=0.8
 10 CONTINUE
 DO 20 I=1,NI+1
 ALPHA=D1*(K-1)
 IFCI.EQ.NI+1) ALPHA=89.995*3.1416/188.
 T=TAN(ALPHA)
 CS=COS(ALPHA)
 AI=T-TCI
 XI1=EXP(-T)/(CS*CS)
 XIR=A1*XI1
 XII=TB1*KXI
 XII=CA1*AI1+(CTB1+TB1)
 VS=XIR/XII
 VS1=XII/XII
 II=L/2*2
 IFCI.EQ.1.OR.I.EQ.NI+1) ID=1
 IFCII.EQ.2) ID=2
 IFCII.NE.1.AND.I.NE.1.AND.I.NE.NI+1) ID=3
 S1(CID)=S1(CID)+VS
 SA1(CID)=SA1(CID)+VS1
 20 CONTINUE
 VAL=CS1(C1)+4.*WS1(C2)+WS1(C3)=D1/3.
 VAI=CSA1(C1)+4.*WSA1(C2)+WSA1(C3)=D1/3.
 45 K=K+1
 S2(C1)=S1(C1)
 S2(C3)=S1(C2)+S1(C3)
 SA2(C1)=SA1(C1)
 SA2(C3)=SA1(C2)+SA1(C3)
 N2=N1
 D2=D1/2.
 S2(C2)=0.8
 SA2(C2)=0.8
 DO 78 I=1,N2

```

ALPHAMOD(2*(K2-1))
T=TANCA(MA)
COS=COSALPHA
AL=AT+C
X1=EXP(-T2/(C3+CD))
X2=(A1+KC1)
X3=(TB1+KC1)
X4=(A1+MA1)+(TB1+TB1)
VS=VTR/VTR
V1=VTR/VTR
S1=C2*(B2CD+VS)
S2(C2)=B2CD+VS
S2(C2)=B2CD+VS1
70 CONTINUE
VAL2=C2*(1+E1+S2(C2)+B2(S3))+D2/3.
VAZ=S2(C1)+E1+B2(C2)+B2(S3))+D2/3.
    DO 100 I=1,2
        X1=EXP(-T2/(C3+CD))
        S1=CD*B2CD
        S2(CD)=B2CD
        CONTINUE
        NH=2*H2
        DH=D2
        IF(K<0,0,100) GO TO 110
        GO TO 45
    C
110 VAL=VAL2
    VAL2=VAZ
    RETURN
    END
    SUBROUTINE BORN(X,T,V1,VAL,V1CD)
    COMPLEX Z,V1,ZI
    VNB=-0.8
    VND=-0.8
    ZHOMPLXT,YT)
    R=ABSSC2
    27 CXT,NE,0,0) TH=TANCA(BS(YT/XT))
    27 CXT,NE,0,0) TH=1416+TH
    27 CXT,NE,0,0,AND,YT,97,0,0) TH=TH+0,0
    27 CXT,NE,0,0,AND,YT,LT,0,0) TH=TH
    27 CXT,LT,0,0,AND,YT,97,0,0) TH=1416+TH
    27 CXT,LT,0,0,AND,YT,LT,0,0) TH=3,1416+TH
    27 CXT,ED,0,0) GO TO 10
    Z=0
    VNB=-0,5772157-CMPLX(CALDBRD,TH-Z)
    DQ=2,B,1868
    20=FLDOTC2
    Z1=CON-1*Z1*(C2/CONX20
    VNB=VNB-Z1
    VND=VND-Z2
    ZHOMPLXT,YT)
    37 CARL,EG,0,0,AND,ADH,EG,0,0) GO TO 25
    BRL=ABSSC1,-VNB/ARL
    B2M=ABSSC1,-VND/AD2O
    IF(CARSBRD,LE,1,E-6,AND,ABSCRD,LE,1,E-6) VNB=VNB*DPCD
    IF(CARSBRD,LE,1,E-6,AND,ABSCRD,LE,1,E-6) GO TO 100
    VNB=VNB

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      VNI=AIN
20    CONTINUE
      VN=VN*EXP(CZ)
      WRITE(6,*)'UNCONVERGENCE Z=',Z
      GO TO 188
      XI=ABSCXT)
      VN=8.5772167-ALOG(XI)-XI
      DO 38 I=2,1000
         XN=FLOAT(I)
         XI=CXH-(D*XI)*ABSCXT)/CXH*XN)
         VN=VN-XI
         ARL=REAL(VN)
         AIM=AIMAG(VN)
         BRL=ABSC1.-ARL/VNR)
         BIM=ABSC1.-AIM/VNR)
         IF(CABS(BRL).LE.1.E-3.AND.ABSCB1D).LE.1.E-3) VN=VN*EXP(CZ)
         IF(CABS(BRL).LE.1.E-3.AND.ABSCB1D).LE.1.E-3) GO TO 188
         VNR=ARL
         VNI=AIN
30    CONTINUE
      TYPE *, 'UNCONVERGENCE X=',X
      VN=VN*EXP(CZ)
100    VALR=REAL(VN)
      VALI=AIMAG(VN)
      RETURN
      END
      SUBROUTINE SUP30(X,Y,Z,ZC,ICP)
      COMMON /A/ NX,NX1,NX2,NZ,NZ1,NZ2,XL,T,XC1D,ZC1D,XC2D,ZC1B
      1,FR,VEL,XS,XE,XB
      COMMON /D/ V1C28,10,33,V2C28,10,33,V3C28,10,33
      COMMON /F/ FC21,11,S3,FI33
      COMMON /A/ Y1,XK,DX1,C1,II
      Y1=Y
      XK=XB
      DO 18 J=1,NZ
         ZI=ZCJ
      DO 18 I=1,NX
         TYPE *, 'SUS J=',J,' I= ',I
         XI=XCJ
         DX1=XX-XI
         C1=ZZ-ZI
         IF(CABS(C1).LE.1.E-6) C1=-1.E-4
         IF(CABS(C1).GT.1.E-6) CALL INT1(F1,ICP)
         IF(CABS(C1).LE.1.E-6) CALL INT2(F1,ICP)
         DO 28 II=1,ICP
            FCI,J,II=F1(II)
28      FCI,J,II=F1(II)
      CONTINUE
10     CONTINUE
      CC=1./3.1415
      DO 38 J=1,NZ
         JI=J+1
      DO 38 I=1,NX1
         II=I+1
      DO 48 II=1,ICP
         P=FC1(J,I,II)+FC1(J,II-FC1,J,I,II-FC1,J,II)
         VSC1(J,II)=P*CC
48      CONTINUE
38      CONTINUE
      RETURN
      END
      SUBROUTINE INT2(F1,ICP)
      DIMENSION F1(S3)

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COMMON /A1/ Y,XX0,DX1,C1,II
EXTERNAL FU
DO 5 II=1,ICP
  IF(DXI.LE.0.0) F1(IXD)=0.0
  IF(CM1.LE.0.0) GO TO 5
  IF(CII.EQ.2) F1(IXD)=0.0
  IF(CII.EQ.2) GO TO 5
  IF(DXI.LE.0.0) DXI=0.0
  SUM=0.0
  SUM1=0.0
  TH1=0.0
  DO 10 I=1,180
    IF(CII.EQ.1) SD=FLOAT(I)
    IF(CII.NE.1) SD=FLOAT(I)/2.
    S=XX0+DX1/(COS(X1/1416))
    IF(S.GT.1.0) SD TO 10
    TH2=ACOS(S)
    IF(TH2.GT.PI/2.) TH2=PI/2.-TH2
    G1=TH1*180./PI/1416
    G2=TH2*180./PI/1416
    CALL SIMDSPN(TH1,TH2,FU,VAL)
    SUM=SUM+VAL
    IF(TH2.GE.PI/2.) GO TO 20
    IF(CSUM.EQ.0.0) CX=0.0
    IF(CSUM.NE.0.0) CX=ABS(SUM1/SUM-1.)
    IF(CX.LE.1.E-4) GO TO 20
    SUM1=SUM
    TH1=TH2
  CONTINUE
  AND=TH2/3.*1416*180.
  F1(IXD)=SUM1.
5   CONTINUE
  RETURN
END
SUBROUTINE INT1(F1,ICP)
DIMENSION F1(99),TC180
COMMON /A1/ Y,XX0,DX1,C1,II
EXTERNAL FU1
FX0=X1*180./PI/1416
GBC(A,B,THET)=A*COS(THETA)-B*TAN(THETA)
PI=3.1416
B=DX1/Y
R=SQR(DXI)=DX1+Y*Y
ALPHA=A*COS(DXI)/R
AA=F1(TH1)
THF=PI/2.
DO 100 III=1,ICP
  TYPE "*****III*****"
  F1(IXD)=0.8
  THI=PI/2.+ALPHA
  TYPE "(*THI*,THI",
  IF(TH1.GT.0.0) GO TO 20
  TT=F1(TH1)
  TYPE "(*,TT,THI*)",TT
  SUM1=0.0
  N=0
  THETA=-68.0*1416/180.
100  N=N+1
  BB=FLOAT(N)
  A=BB*PI/2.*COS(X1)
  TH2=ATAN(A*COS(THETA)-B)
  IF(CABS(THETA/TH2-1.0).LE.1.E-2) TS=(THETA+TH2)/2.
  
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IF(CABS(THETA/TH2-1.) LE .1.E-2) CALL STACKCN, TS, TD
IF(CABS(THETA/TH2-1.) LE .1.E-2) GO TO 18
THETA=TH2+THETA/2.
IF(THETA.LT.0.0) GO TO 5
N=N+1
C      TYPE N, 'THE NUMBER OF ZERO POINT N', N
IF(N.EQ.0) N=1
IF(N.GT.100) NN=100
IF(N.LE.100) NN=N
AU=0.8
DO 15 I=1,NN
  NI=NN-I+1
  AL=TCNI
  IF(CAL.LT.TH1) AL=TH1
  IF(CN.EQ.1) AL=TH1
  CALL SIMPSNCAL,AU,FUI,VALD
  A1=FCAD0
  A2=FCAL0
C      TYPE N, 'AU=', A1, 'AL=', A2, 'VAL=', VAL
  F1(CID)=F1(CID)+VAL
C      TYPE N, 'F1(CID)=', F1(CID)
  IF(CABS(TH1).LE.1.E-6) GO TO 20
  IF(F1(CID).EQ.0.0) CX=0.8
  IF(F1(CID).NE.0.0) CX=ABS(SUM1/F1(CID)-1.)
  IF(CX.LE.1.E-4) GO TO 20
  SUM1=F1(CID)
  AU=AL
CONTINUE
15    IF(TH1.LT.0.0) TH1=-0.8
XL=TH1
XL=TH1
XU=89.9993.1416/188.
BBB=FLOATND
A=BBB*PI/(CXK0*Y)
XH=CXL+XU)/2.
YL=88CA.B,XLD
YU=88CA.B,XUD
YH=88CA.B,XHD
TYPE N, 'XL=', XL, 'YL=', YL
TYPE N, 'XH=', XH, 'YH=', YH
TYPE N, 'XU=', XU, 'YU=', YU
IF(CYH>YL) GE. 8.0) XL=XH
IF(CYH>YL) LT. 8.0) XU=XH
T1=FCXL0
T2=FCXU0
IF(CABS(XU-XLD).LE.1.E-6) TH2=(XL+XU)/2.
IF(CABS(XU-XLD).LE.1.E-6) GO TO 38
GO TO 25
30    IF(TH2.LT.TH1) GO TO 50
C      IF(CABS(TH1-TH2).LE.1.E-6) TH2=89.9993.1416/188.
CALL SIMPSNC(TH1, TH2, FUI, VALD)
A1=FCTH1
A2=FCTH2
C      TYPE N, 'TH1=', A1, 'TH2=', A2, 'VAL=', VAL
  F1(CID)=F1(CID)+VAL
  IF(TH1.0T.FC880.AND.ABS(VALD).LE.1.E-20) GO TO 100
  TYPE N, 'F1(CID)=', F1(CID)
  IF(F1(CID).EQ.0.0) GO TO 66
  IF(F1(CID).NE.0.0) CX=ABS(SUM1/F1(CID)-1.)
  IF(CX.LE.1.E-4) GO TO 100
  SUM1=F1(CID)
  TH1=TH2
  N=N+1

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XL=TH1
IFCN.LE.1000) GO TO 23
188 CONTINUE
78 RETURN
END
SUBROUTINE STACK(N,TS,T)
DIMENSION T(10),
IFCN.LE.10) GO TO 10
DO 20 I=1,9
  II=I+1
  T(I)=T(II)
20 CONTINUE
T(10)=TS
GO TO 38
18 T(10)=TS
RETURN
END
SUBROUTINE SIMPSN(A,B,F,VAL)
DIMENSION S1(3),S2(3)
NI=18
DI=(B-A)/NI
KI=1
DO 18 I=1,3
  S1(I)=0.8
18 CONTINUE
DO 28 I=1,NI+1
  ALPHA=A+DI*(I-1)
  VS=F(CALPHA)
  VS=F(CALPHA)
  II=I/2#2
  IF CI.EQ.1.OR.I.EQ.NI+1) IO=1
  IF CI.EQ.1) IO=2
  IF CI.EQ.1.AND.I.NE.1.AND.I.NE.NI+1) IO=3
  S1(CI)=S1(CI)+VS,
28 CONTINUE
VAL1=S1(1)+4.*S1(2)+2.*S1(3))*DI/3.
45 KKI=1
  S2(1)=S1(1)
  S2(3)=S1(2)+S1(3)
  N2=NI
  D2=DI/2.
  S2(2)=0.8
  DO 78 I=1,N2
    ALPHA=A+D2*(2*I-1)
    VS=F(CALPHA)
    VS=F(CALPHA)
    S2(2)=S2(2)+VS
78 CONTINUE
VAL2=(S2(1)+4.*S2(2)+2.*S2(3))*D2/3.
IFCVAL1.EQ.8.80 CX=-8.8
IFCVAL1.NE.8.80 CX=ABSCVAL2/VAL1-1.0
IFCX.LE.1.E-40 GO TO 118
VAL1=VAL2
DO 100 J=1,3
  S1(CJ)=S2(CJ)
100 CONTINUE
NI=2*N2
DI=D2
IFCK.GT.10) GO TO 118
GO TO 45
C
118 VAL=VAL2
200 RETURN
END

```

```
FUNCTION FUCHETAJ
DOUBLE PRECISION XI1,F1,F2,B1,B2,A
COMMON /AI/ Y,X0B,DX1,C1,I
F1=COS(CTHETAJ)
F2=SIN(CTHETAJ)
A=X0B/(CF1*F1)
XI1=EXP(CAMC1D)
B1=X0B*DX1/F1
B2=A*F2*Y
IFCI.EQ.1) FU=XI1*SIN(B1)
IFCI.EQ.2) FU=XI1*COS(B1)*F2/F1
IFCI.EQ.3) FU=XI1*(COS(B1)-1.0)/F1
IFCI.EQ.5) FU=XI1*COS(B1)/F1
RETURN
END
FUNCTION FUI(CTHETAJ)
COMMON /AI/ Y,X0B,DX1,C1,I
F1=COS(CTHETAJ)
F2=SIN(CTHETAJ)
A=X0B/(CF1*F1)
XI1=EXP(CAMC1D)
B1=A*(DX1*F1+Y*F2)
IFCI.EQ.1) FUI=XI1*SIN(B1)
IFCI.EQ.2) FUI=XI1*SIN(B1)*F2/F1
IFCI.EQ.3) FUI=XI1*(COS(B1)-1.0)/F1
RETURN
END
```

